



**GVK(KSF SERIES)
PRESSURE SAFETY VALVE**
AUTOMATION & REGULATING VALVES

**AUTOMATION &
REGULATING VALVES**
Total Engineering Solution Service

GvK Limited

HEAD OFFICE

Techno Valley C-503, 236, Jubuto-ro,
Bupyeong-gu, Incheon, Republic of Korea

TEL +82-32-678-7972
FAX +82-32-678-7973
E-MAIL ican@gvkvalve.com

www.gvk.co.kr
www.gvkvalve.com





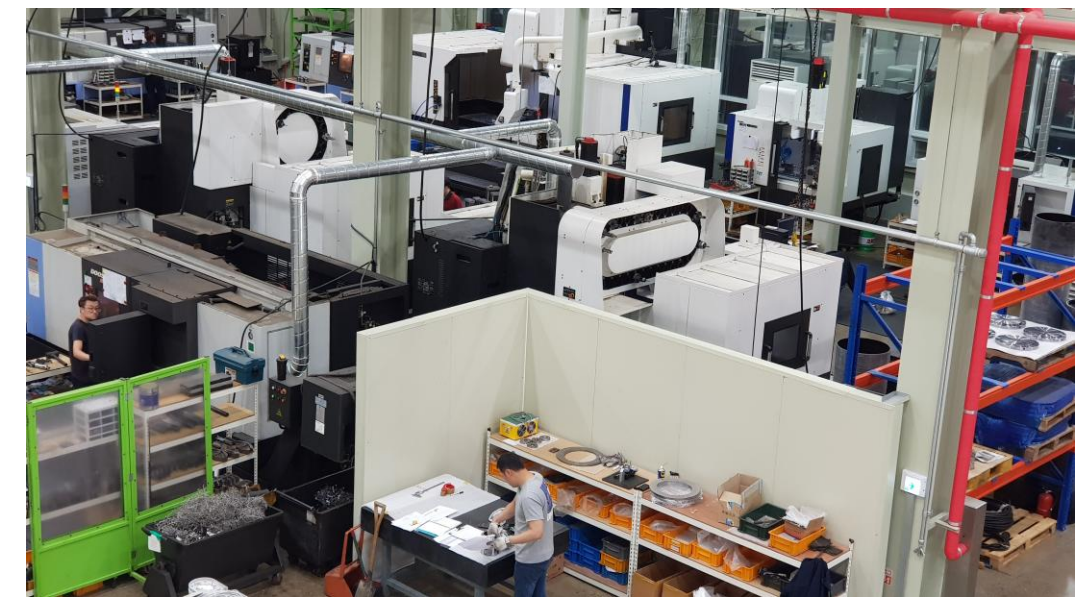
GvK Limited, founded in June 2020 by a leader with 38 years of experience, focuses on R&D while ensuring quality, price, and functionality through domestic production. The company offers Process Valves, Valve Equipment, and Total Engineering services for industries such as Gas, Refining, Petroleum, Power generation, Environment, and Water treatment. With a management team possessing 30-40 years of experience, GVK has developed numerous patents and adheres to quality standards like ISO 9001, 14001, 45001, and CE, ASME "U" & "PP", EAC. Recognized for its advanced automatic control valves, GVK also supplies a range of Control Valves globally through OEM and ODM partnerships.

Although Still In The Early Design And Manufacturing Stages, GVK Has Emerged As A Leading Company In Korea, Equipped With Skilled Personnel And Testing Capabilities. The Company Aims To Lower Production Costs, Enhance Efficiency And Improve Quality While Accumulating Hyper-intelligence Valve Engineering (Hive) Technology. GvK Limited Is Committed To Meeting Customer Needs With Competitive Pricing And High Value-added Services.

Mutually Beneficial Relationships



To Be Your Best Partner



The Professional Provider of Automatic valve Actuators



38 years experience "one stop" goods and services





Mission

기초에 근거하여 원천 기술을 살리고 원천 기술을
극대화하여 세계 중심에 서는 것

To place oneself at the center of the global landscape by utilizing
core groundbreaking technology and maximizing its capabilities.

Vision

우리는 옳은 일과 가치 있는 일에 주저함이 없이
최선을 다하고 실천하여 세계의 중심에 서자

Let us place ourselves at the forefront of the world by striving to do
our best and confidently advocating for what is just and valuable.

Contents

01	DESIGN FUNDMENTALS
04	TERMINOLOGY FOR PRESSURE RELIEF DEVICES
07	PRESSURE LEVELS
09	KS B 6216 INLET FLANGE DIMENSION
	DIMENSION FOR BOILER
10	BILL OF MATERIALS - CONVENTIONAL
12	BILL OF MATERIALS – BELLOWS
13	TOP CONSTRUCTION AND TYPE
14	GENARAL DIMENSION
17	STANDARD VALVE SELECTION TABLES
18	STANDARD VALVE SELECTION TABLES(ASME)
20	STANDARD VALVE SELECTION TABLES(SEC)
22	CALCULATION OF SAFETY VALVE CAPACITY
23	SAFETY VALVE CAPACITY TABLE
25	SUPERHEAT CORRECTION FACTOR
	FOR SAFETY VALVES
31	CALCULATION OF SAFETY VALVE CAPACITY
41	MAINTENANCE MANUAL
47	BILL OF MATERIALS
51	GENARAL DIMENSION
52	STANDARD VALVE SELECTION TABLE
53	CALCULATION OF SAFETY & RELIEF VALVE CAPACITY
57	NOTE
61	CERTIFICATE

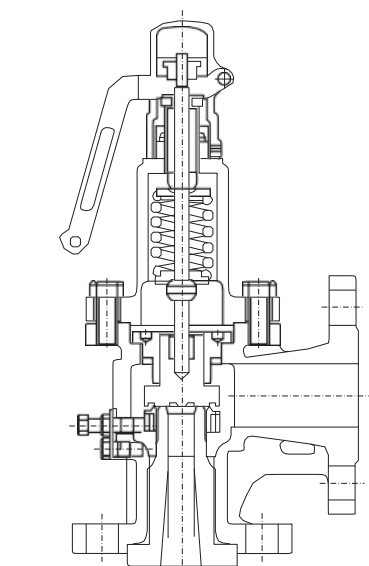
DESIGN FUNDAMENTALS

Introduction

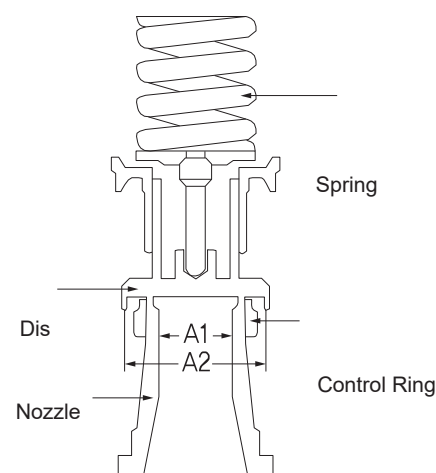
A pressure relief valve is a safety device designed to protect a pressurized vessel or system during an overpressure event. An overpressure event refers to any condition which would cause pressure in a vessel or system to increase beyond the specified design or maximum allowable working pressure. Since pressure relief valves are safety devices, there are many Codes and Regulations written to control their design and application. The purpose of this discussion is to familiarize you with the various parameters involved in the design of a pressure relief valve and provide brief introduction to some of the Codes and Standards which govern the design and use of pressure relief valves. Many electronic, pneumatic, and hydraulic systems exist today to control fluid system variables, such as pressure, temperature, and flow. Each of these system requires a power source of some type, such as electricity or compressed air in order to operate. A pressure relief valve must be capable of operating at all times, especially during a period of power failure when system controls are nonfunctional. The sole source of power for the pressure relief valve, therefore, is the process fluid. Once a condition occurs, which causes the pressure in a system or vessel to increase to a dangerous level, the pressure relief valve may be the only device remaining to prevent a catastrophic failure. Since reliability is directly related to the complexity of the device, it is important that the design of the pressure relief valve be as simple as possible. The pressure relief valve must open at a predetermined set pressure, flow a rated capacity at a specified overpressure, and shut off when the system pressure has returned to a safe level. Pressure relief valves must be designed with materials compatible with all process fluids from simple air and water to the most corrosive media. With means to derive maximum fluid capacity for a given pipe size, and with consistently smooth stable operation on all fluids and fluid phases.

Spring Loaded Design

The basic Spring Loaded pressure relief valve has been developed to meet the need for a simple, reliable system actuated device to provide overpressure protection. Figure 1 shows the construction of a spring loaded pressure relief valve. The valve consists of a valve inlet or nozzle mounted on the pressurized system, a disc held against the nozzle to prevent flow under normal system operating conditions, a spring to hold the disc closed, and a housing to contain the operating elements. The spring is adjustable to vary the pressure at which the valve will open. Figure 2 is a simple sketch showing the disc held in the closed position by the spring. When system pressure reaches the desired opening pressure, the force of pressure acting over Area A1 equals the force of spring and the disc will lift and allow fluid to flow out through the valve. When pressure in the system returns to a safe level, the valve will reclose. When a pressure relief valve begins to lift, the spring force increases. This means that system pressure must increase if lift is to continue. For this reason pressure relief valves are allowed an overpressure allowance to reach full lift. This allowable overpressure can vary from 3% for valves on fired vessels to 10% for valves on unfired systems. This margin is relatively small and some means must be provided to assist in the lift effort. Most pressure relief valves, therefore, have a secondary control chamber or huddling chamber to enhance lift. A typical configuration is shown in Figure 3. As the disc begins to lift, fluid enters the control chamber exposing a larger area of the disc to system pressure A2(Figure 2). This causes an incremental change in force which overcompensates for the increase in spring force and causes the valve to open at a rapid rate. At the same time, the direction of the fluid flow is reversed and the momentum effect resulting from the change in flow direction further enhances lift. These effects combine to allow the valve to achieve maximum lift and maximum flow within the allowable overpressure limits.

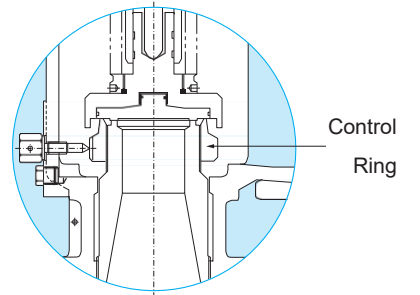


Spring Loaded Pressure Relief Valve
Figure 1



Valve Trim Diagram
Figure 2

Because of the larger disc Area A2(Figure 2) exposed to system pressure after the valve achieves lift, the valve will not shut off until system pressure has been reduced to some level below the set pressure. The design of the control Chamber determines where the shut off will occur. The difference between the set pressure and the shut off pressure is called blowdown and is usually expressed as a percentage of set pressure. The required blowdown can vary from 4% for valves on fired vessels to indeterminate values for valves on unfired systems. The design of the control or huddling chamber involves a series of design trade offs. If the design maximizes lift effort then blowdown will be long. If the design objective is to minimize blowdown, then the lift effort will be diminished. Many pressure relief valves are, therefore, equipped with a control ring which can be adjusted to vary the geometry of the control chamber to meet a particular system operating requirement(Figure 2 & 3). Compatibility with the process fluid is achieved by careful selection of materials of construction. Materials must be chosen with sufficient strength to withstand the pressure and temperature of the system fluid. Materials must also resist chemical attack by the process fluid and the local environment to insure valve function is not impaired over long periods of exposure. Bearing properties are carefully evaluated for parts with sliding surfaces and the ability to achieve a fine finish in the seating surfaces of the disc and nozzle is required for tight shut off.



Pressure Relief Valve Trim
Figure 3

Back Pressure Considerations

In the past, pressure relief valves were generally vented to atmosphere. As processes have become more sophisticated and the emission of hazardous materials has become a widespread concern, pressure relief valves are required to vent into closed systems. When valves are allowed to vent to atmosphere, the bonnet of the valve may be opened. This insures that pressure in the outlet of the valve is normally atmospheric and protects the spring from the harmful effects of high temperature fluid. With closed system applications, the bonnet must be totally enclosed. This introduces the possibility that back pressure will exist in the outlet of the valve. A review of the force balance on the disc(Figure 2) shows that the force of fluid pressure acting on the inlet side of the disc will be balanced by the force of the spring plus whatever pressure exists on the outlet side of the valve. If pressure in the valve outlet varies while the valve is closed, the valve set pressure will change. If back pressure varied while the valve is open and flowing, valve lift and flow rate through the valve can be affected. Care must be taken in the design and application of pressure relief valves to compensate for these variations by simply changing the disc insert in a matter of minutes with minimal conversion parts. Back pressure which may occur in the valve outlet system while the valve is closed is called static or superimposed back pressure. This back pressure may be a result of the valve outlet being connected to a normally pressurized system or may be caused by other pressure relief valves venting into a common header. Compensation for superimposed back pressure which is constant may be provided by reducing the spring force. Under this condition the force of the spring plus back pressure acting on the disc would equal the force of the inlet set pressure acting to open the disc. It must be remembered, however, that the valve of the set pressure will vary directly with any change in back pressure. When superimposed back pressure is variable, a balanced bellows design is recommended. A typical balanced bellows style valve is shown in Figure 4. The bellows is designed with an effective pressure area equal to the seat area of the disc. The bonnet is vented to insure that the pressure area of the bellows will always be exposed to atmospheric pressure and to provide a tell-tale should the bellows begin to leak. Variations in back pressure, therefore, will have no effect on set pressure. Back pressure may, however, affect flow. Back pressure, which may occur after the valve is open and flowing, is called dynamic or built up back pressure. This type of back pressure may be caused by fluid flowing from the pressure relief valve into the downstream piping system. Developed back pressure will not affect the valve opening pressure, but may have an effect on valve lift and flow. Where developed back pressure is expected to exceed 10% of set pressure, a balanced bellows design is recommended.

Nozzle Type

The inlet construction of pressure valves is either a full nozzle or semi nozzle. In a full nozzle valve, only the nozzle and disc are exposed to the fluid media when the valve is closed. In a semi-nozzle valve the nozzle, disc, and part of the valve body are exposed to the inlet fluid when the valve is closed. Semi nozzles are usually welded into the body and cannot be easily removed.

Balanced Bellows

In addition to offsetting the effects of variable back pressure, the bellows acts to seal process fluid from escaping to atmosphere and isolates the spring, Bonnet, and guiding surface from contacting the process fluid. This is especially important for corrosive services.

Seat Leakage

Another important consideration in the design on a pressure relief valve is the ability to maintain tight shut off. Pressure relief valves are required to remain on systems for long periods of time under widely varying conditions of pressure and temperature. Seat leakage will result in continuous loss of system fluid and may cause progressive damage to the valve seating surfaces. Extreme leakage could result in premature opening of the valve. Allowable seat leakage limits for pressure relief valves are, therefore, many orders of magnitude more stringent than required for other types of valves. These extremes of tightness are achieved by close control of part alignment, optically flat seating surfaces, and careful selection of materials for each application. A diligent maintenance schedule must be carried out in the field to maintain the leak tight integrity of the valve, particularly on a system where the pressure relief valve is cycled often. For additional tightness, where system conditions permit, soft seat or elastomer seat construction may be employed. Most manufactures recommend that system operating pressure not exceed 90% of set pressure to achieve and maintain proper seat tight integrity.

Soft Seat

O- ring soft seat seals provide positive closure at service pressures closer to the set pressure than is possible with metal- to-metal seats assuring continuos, trouble-free service, and complete valve tightness after numerous pops.

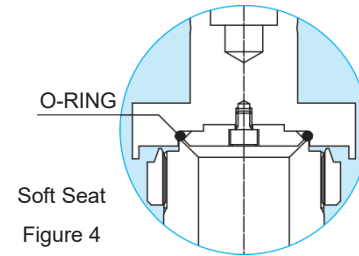


Figure 4

Code and Standard

Many Codes and Standards are published throughout the world which address the design and application of pressure relief valves. The most widely used and recognized of these is the ASME Boiler and Pressure Vessel Code, commonly called the ASME Code. Most Codes and Standards are voluntary which means that they are available for use by manufacturers and users and may be written into purchasing and construction specifications. The ASME Code is unique in the United States and Canada in that it has been adopted by the majority of state and provincial legislatures and is mandated by law. The ASME Code provides rules for the design and construction of pressure vessels. Various sections of the Code cover fired vessels, nuclear vessels, unfired vessels, and additional subjects, such as, welding and nondestructive examination. Vessels manufactured in accordance with the ASME Code are required to have overpressure protection. The type and design of allowable overpressure protection devices is spelled out in detail in the Code. The ASME Code also provides specific rules governing the application of overpressure protection, determination of and allowable tolerance on set pressure, allowable overpressure, required blowdown, application of multiple valves, sizing for fire, requirements for materials of construction, and rules for installation. The most widely used pressure relief valve voluntary standards in the United States are published by the American Petroleum Institute(API). These Standards provide recommendations for pressure relief valve construction, sizing, installation, and maintenance. The API, more than any other body, has worked to standardize the ratings and size of pressure relief valves. API developed a series of inlet, orifice, outlet combinations for various flanged valve pressure classes which are utilized throughout the petroleum and process industry. These standard sizes are characterized by a series of fourteen(14) standard letter orifices ranging from D through T. Each letter refers to a specific effective orifice area. This orifice area is used in standard API formulations to calculate valve flow rate. The manufacturer is not required to produce a valve with a bore area equal to the effective area. Rather, he is obliged to produce a valve which will have a flow rate equal to or greater than that determined by the API formulation. Many other Standards are published which deal with the application and design of pressure relief valves peculiar to a specific industry. Additional Codes and Standards are written by various bodies throughout the world. A group is currently working within the International Standards Organization(ISO) to create an International Standard for pressure relief valve.

Sizing Pressure Relief Valve

The first step in applying overpressure protection to a vessel or system is to determine the set pressure allowable overpressure and required relieving capacity. The set pressure and allowable overpressure can be easily determined by reference to the operating pressure of the system and Code under which the system or vessel will be built and inspected. The most difficult task is in determining the required relieving capacity. The pressure relief valve must relieve a sufficient amount of fluid to insure that pressure in the vessel or system never exceeds the specified overpressure. This means that all possible sources and causes of overpressure must be evaluated. Some examples could be failure of a stop valve to close, control system failure, fire, pump failure, uncontrolled chemical reaction, vessel isolation, and many more. The worst case combination of these factors is used to determine the required capacity. Once these factors have been determined, the pressure relief valve is chosen so that the rated relieving capacity of the valve exceeds the required capacity derived from the worst case system failure analysis.

Summary

The purpose of this discussion has been to provide an introduction to some of the design considerations employed when designing a pressure relief valve and to the Codes and Standards which are employed in this industry to maintain a high level of product quality and reliability. More specific information may be found by referencing the ASME Code, various published Standards and by consulting literature published by the pressure relief valve manufactures. A key point to keep in mind is the pressure relief valve is a safety device employed to protect a pressure vessel or system from catastrophic failure. With this in mind, the application of pressure relief valves should be assigned only to fully trained personnel and rules provided by the governing Codes should be strictly complied with.

1. Scope

The Scope of this standard is to define pressure relief devices, their functional and operational characteristics, and to standardize the terminology covering such devices and their characteristics. These devices are intended to protect containers and equipment from abnormal internal or external pressure.

2. General

2.1 Pressure Relief Devices

A pressure relief devices is designed to open to prevent a rise of internal fluid pressure in excess of a specified valve due to exposure to emergency or abnormal conditions. It may also be a pressure relief valve, a non-reclosing pressure relief devices or a vacuum relief valve.

3. Type of Devices

3.1 Pressure Relief Valve

A pressure relief valve is a pressure relief device which is designed to reclose and prevent the further flow of fluid after normal conditions have been restored.

3.1-1 Safety Valve

A safety valve is a pressure relief valve actuated by inlet static pressure and characterized by rapid opening or pop action.

3.1-2 Relief Valve

A relief valve is a pressure relief valve actuated by inlet static pressure which opens in proportion to the increase in pressure over the opening pressure.

3.1-3 Safety Relief Valve

A safety relief valve is a pressure relief valve characterized by rapid opening or pop action, or by opening in proportion to the increase in pressure over opening pressure, depending on application.

3.1-4 Conventional Safety Relief Valve

A safety relief valve is a pressure relief valve characterized by rapid opening or pop action, or by opening in proportion to the increase in pressure over opening pressure, depending on application.

3.1-5 Balanced Safety Relief Valve

A balanced safety relief valve incorporate means of minimizing the effect of back pressure on the operational characteristics(opening pressure, closing pressure and relieving capacity)

4. Parts of Pressure Relief Devices

4-1. Approach Channel

An approach channel is the passage through which the fluid must pass to reach the operating parts of a pressure relief device.

4-2. Discharge Channel

A discharge channel is the passage through which the fluid must pass beyond the operating parts of a pressure relief device.

4-3. Disc

A disc is the pressure containing movable element of a pressure relief valve which effects closure.

4-4. Lifting Device

A lifting device is a device for manually opening a pressure relief valve by the application of external force to lessen the spring loading which holds the valve closed.

4-5. Lifting Lever

See Lifting Device.

4-6. Nozzle

A nozzle is the pressure containing element which constitutes the inlet flow passage and includes the fixed portion of seat closure.

4-7. Seat

A seat is the pressure containing contact between the fixed and moving portions of the pressure containing elements of a valve.

5. Pressure Relief Valve Dimensional Characteristics

5-1. Actual Discharge Area

Actual discharge area is the measured minimum net area which determines the flow through a valve.

5-2. Bore Area

Bore area is the minimum cross sectional area of the nozzle.

5-3. Bore Diameter

Bore diameter is the minimum diameter of the nozzle.

5-4. Curtain Area

Curtain Area is the area of the cylindrical or conical discharge opening created between the seating surfaces by the lift of the disc above the seat.

5-5. Developed Lift

The developed lift is the actual travel of the disc from closed position to the position reached when the valve is at flow rating pressure.

5-6. Effective Discharge Area

Effective discharge area is a normal or computed area of flow through a pressure relief valve, differing from the actual discharge area, for use in recognized flow formulas to determine the capacity of a pressure relief valve.

5-6. Inlet Size

Effective discharge area is a normal or computed area of flow through a pressure relief valve, differing from the actual discharge area, for use in recognized flow formulas to determine the capacity of a pressure relief valve.

6. Operational Characteristics of Pressure Relief Valve

6-1. Back Pressure

Back pressure is the static pressure existing at the outlet of a pressure relief device due to pressure in the discharge system.

6-2. Blowdown

Blowdown is the difference between actual popping pressure of a pressure of a pressure relief valve and actual reseating pressure expressed as a percentage of set pressure units.

6-3. Blowdown Pressure

Blowdown pressure is the value of decreasing inlet static pressure at which no further discharge is detected at the outlet of a safety relief valve of the resilient disc type after the valve has been subjected to a pressure equal to or above the popping pressure.

6-4. Built-up Back Pressure

Built-up back pressure is pressure existing at the outlet of a pressure relief device occasioned by the flow through the particular device into a discharge system.

6-5. Closing Pressure

Closing pressure is the value of decreasing inlet static pressure at which the valve disc reestablished contact with the seat or at which lift becomes zero.

6-6. Coefficient of Discharge

Coefficient of discharge is the ratio of the measured relieving capacity to the theoretical relieving capacity.

6-7. Cold Differential Test Pressure

Cold differential test pressure is the inlet static pressure at which a pressure relief valve is adjusted to open on the test stand. This test pressure includes corrections for service condition of back pressure and or temperature.

6-8. Flow Rating Pressure

Flow rating pressure is the inlet static pressure at which the relieving capacity of a pressure relief device is measured for rating purposes.

6-9. Start-to-discharge Pressure

Start-to-discharge pressure is the value of increasing inlet static pressure at which the first bubble occurs when a safety relief valve of the resilient disc type is tested by means of air under a specified water seal on the outlet.

6-10. Leak Test Pressure

Leak test pressure is the specified inlet static pressure at which a quantitative seat leakage test is performed in accordance with a standard procedure.

6-11. Measured Relieving Capacity

Measured relieving capacity is the relieving capacity of a pressure relief device measured at the flow rating pressure, expressed in gravimetric or volumetric units.

6-12. Opening Pressure

Opening pressure is the value of increasing inlet static pressure of a pressure relief valve at which there is a measure lift, or at which the discharge becomes continuous as determined by seeing, feeling, or heating.

6-13. Overpressure

Overpressure is a pressure increase over the set pressure of a pressure relief valve, usually expressed as a percentage of a set pressure.

6-14. Popping Pressure

Popping pressure is the value of increasing inlet static pressure at which the disc moves in the opening direction at a faster rate as compared with corresponding movement at higher or lower pressures. It applies only to safety or safety relief valves on compressible fluid service.

6-15. Rated Relieving Capacity

Rated relieving capacity is that portion of the measured relieving capacity permitted by the applicable code or regulation to be used as a basis for the application of a pressure relief device.

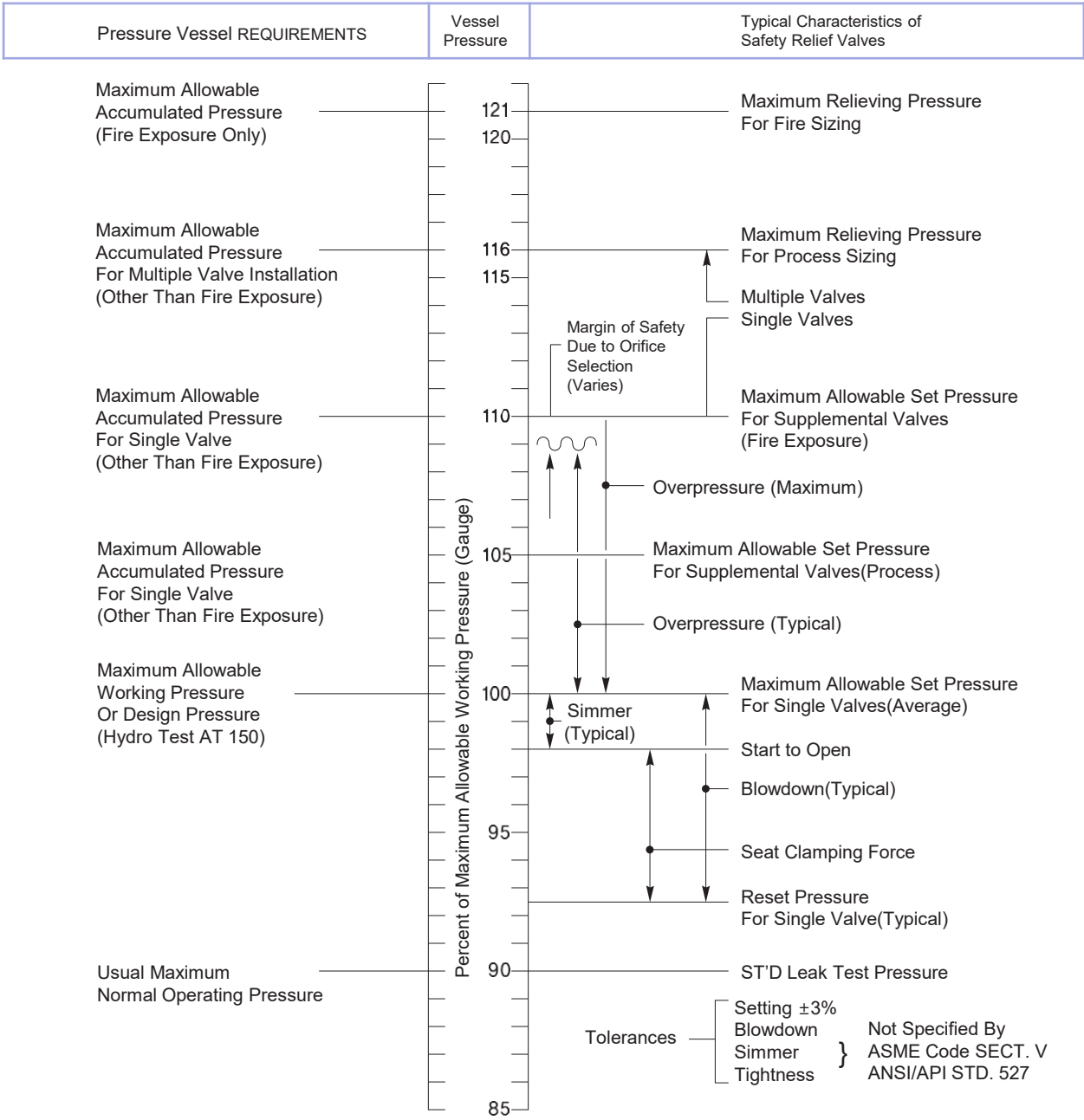
6-16. Relieving Pressure

Relieving pressure is set pressure plus overpressure.

6-17. Set Pressure

Set pressure is the value of increasing inlet static pressure at which a pressure relief valve displays one of the operational characteristics as defined under opening pressure popping pressure or start-to- discharge pressure depending on service or as designated by the applicable code or regulation. It is one value of pressure stamped on the pressure relief valve.

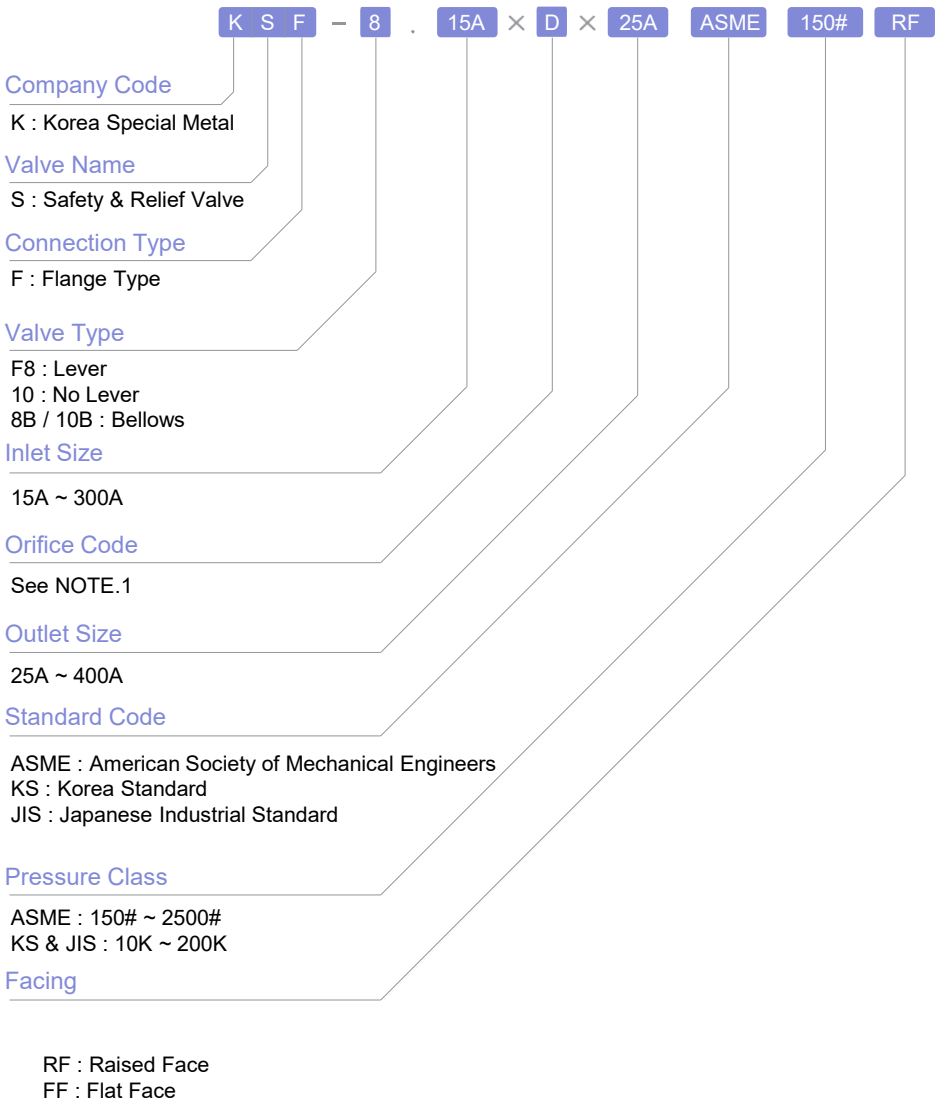
PRESSURE LEVELS



- Operating pressure may be any lower pressure required.
- Set pressure, and all other values related to it may be moved downward if operation pressure permits.
- This chart is in accordance with the requirements of the ASME Boiler and Pressure Vessel Code, Sect. VIII Div. 1 Pressure Vessels.

Numbering System

Model	Type	Lever	Use Condition
KSF – 8	Conventional	Yes	Steam, Air, Gas, Vapor
KSF – 8B	Bellows	Yes	Steam, Air, Gas, Vapor, Back Pressure
KSF – 10	Conventional	No	Steam, Air, Gas, Vapor, Liquid
KSF – 10B	Bellows	No	Steam, Air, Gas, Vapor, Liquid, Backpressure

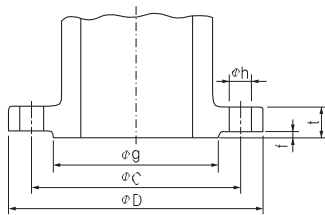


Note. 1 / Orifice Code

Code	D	e	f	g	h	j	k	l	m	n	p	q	r	t	v	w
Orifice	10	13	16	21	26	33	39	49	55	60	73	96	115	150	204	238

KS B 6216 INLET FLANGE DIMENSION FOR BOILER

BILL OF MATERIALS - CONVENTIONAL



KS B 6216 Inlet Flange Dimension

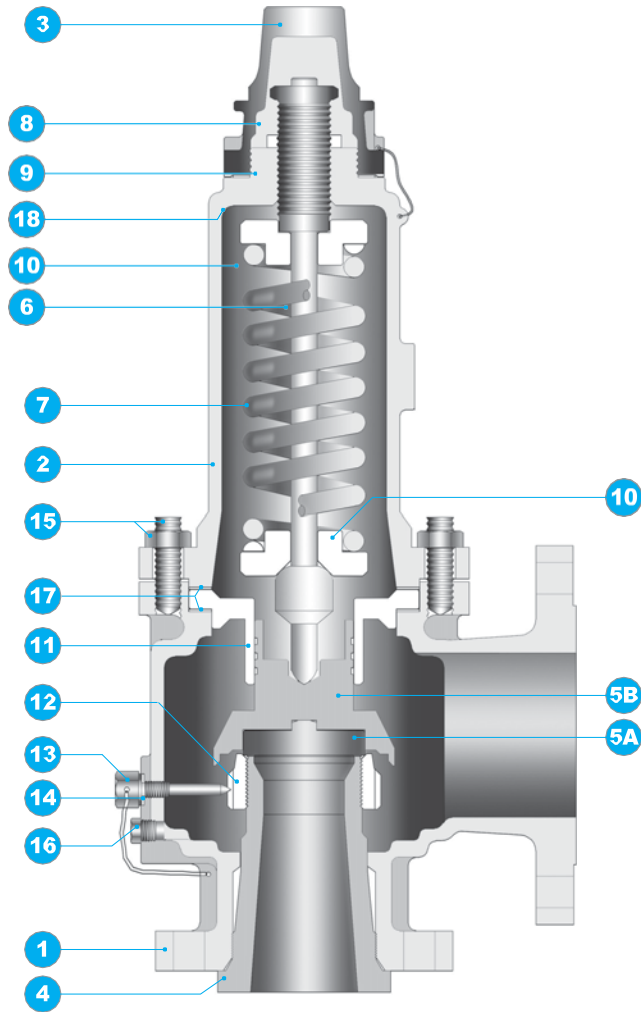
Nominal Pressure	Nominal Diameter	Flange				Bolt Hole			Bolt
		D	t (min)	f	g	C	Number	h	
10K	20	125	22	1	56	90	4	19	M16
	25	130	22	1	67	95	4	19	M16
	32	140	24	2	76	105	4	19	M16
	40	155	24	2	81	120	8	19	M16
	50	165	26	2	96	130	8	19	M16
	65	200	28	2	116	160	8	23	M20
	80	210	30	2	126	170	8	23	M20
	-90	225	30	2	136	185	8	23	M20
	100	245	32	2	151	205	8	23	M20
	125	280	34	2	182	235	12	25	M22
20K	150	325	36	2	212	280	12	25	M22
	200	385	38	2	262	335	12	27	M24
	20	130	22	1	56	95	4	19	M16
	25	135	22	1	67	100	4	19	M16
	32	160	24	2	76	120	4	23	M20
	40	165	24	2	81	130	8	19	M16
	50	185	26	2	96	145	8	23	M20
	65	210	30	2	116	170	8	23	M20
	80	230	32	2	132	185	8	25	M22
	-90	240	34	2	145	195	8	25	M22
30K	100	265	36	2	160	220	8	25	M22
	125	290	38	2	195	245	12	25	M22
	150	350	42	2	230	300	12	27	M24
	200	410	46	2	275	350	12	33	M30
	20	130	24	1	60	95	4	19	M16
	25	135	24	1	70	100	4	19	M16
	32	160	26	2	80	120	4	23	M20
	40	165	28	2	90	130	8	19	M16
	50	185	30	2	105	145	8	23	M20
	65	210	34	2	130	170	8	23	M20
45K	80	230	36	2	140	185	8	25	M22
	-90	250	38	2	150	205	8	25	M22
	100	285	40	2	160	235	8	27	M24
	125	315	44	2	195	265	12	27	M24
	150	375	48	2	235	315	12	33	M30
	200	435	54	2	280	370	12	36	M33
	20	140	30	1	60	100	4	23	M20
	25	150	30	1	65	110	4	23	M20
	32	175	34	2	75	130	4	25	M22
	40	185	36	2	90	145	8	23	M20
65K	50	205	38	2	100	160	8	25	M22
	65	230	42	2	120	185	8	25	M22
	80	255	44	2	130	205	8	27	M24
	-90	270	46	2	145	220	8	27	M24
	100	310	50	2	165	250	8	33	M30
	-115	325	52	2	175	265	8	33	M30
	125	335	54	2	195	275	12	33	M30
	150	400	58	2	235	335	12	33	M30
	200	460	66	2	285	390	12	39	M36
	20	149	40	6	51	101.5	4	25	M22
65K	25	159	40	6	63	111	4	25	M22
	32	178	46	6	73	124	4	27	M24
	40	216	52	6	92	165	8	25	M22
	50	244	56	6	105	190.5	8	27	M24
	65	267	62	6	127	203	8	33	M30
	80	311	68	6	157	241.5	8	33	M30
	-90	311	68	6	157	241.5	8	33	M30
	100	375	86	6	186	292	8	39	M36
	-115	375	86	6	186	292	8	39	M36
	125	394	96	6	216	317.5	12	39	M36
65K	150	483	105	6	270	393.5	12	46	M42
	200	584	121	6	324	482.5	12	52	M48

(Unit : mm)

Remark : The nominal diameter within () shall not be used, if possible.

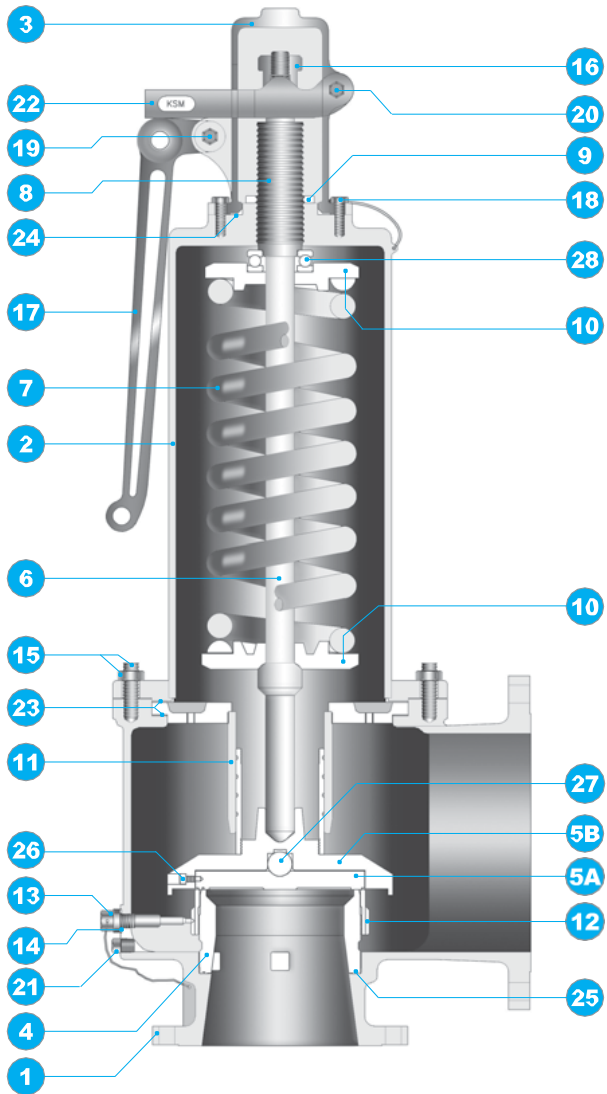
KSF – 10

Size 1/2" (15A) ~ 8" (200A)



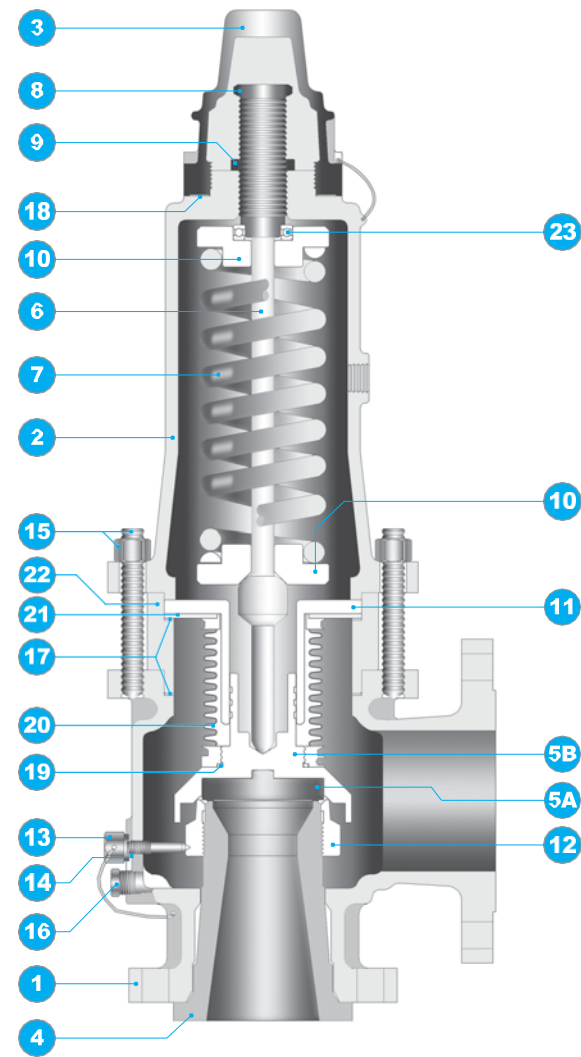
No.	Part Name	Standard Material
1	Body	Ductile
		Carbon steel stainless steel
2	Bonnet	Ductile
		Carbon steel stainless steel
3	Cap	Carbon steel
		Stainless steel
4	Seat	Stainless steel
5A	Disc	Stainless steel
5B	Disc holder	Stainless steel
6	Spindle	Carbon steel
		Stainless steel
7	Spring	Carbon steel
		Stainless steel high. Temp. Alloy
8	Adj. Screw	Bronze
		Carbon steel stainless steel
9	Adj. Screw lock nut	Bronze carbon steel
		Stainless steel
10	Spring guide	Carbon steel
		Stainless steel
11	Disc guide	Brass
		Stainless steel
12	Blow down ring	Brass
		Stainless steel
13	Lock bolt	Carbon steel
		Stainless steel
14	Gasket	Non-asbestos
		Ptfe graphite
15	Stud. Bolts & nuts	Carbon steel
		Stainless steel
16	Drain plug	Brass
		Carbon steel stainless steel
17	Gasket	Non-asbestos
		Ptfe graphite
18	Gasket	Non-asbestos
		Ptfe graphite

KSF Series Semi Nozzle Type
Size 10" (250A) ~ 20"(500A)



No.	Part Name	Standard Material
1	Body	Carbon steel
		Stainless steel
2	Bonnet	Carbon steel
		Stainless steel
3	Cap	Ductile
		Carbon steel stainless steel
4	Seat	Stainless steel
5a	Disc	Stainless steel
5b	Disc holder	Stainless steel
6	Spindle	Carbon steel
		Stainless steel
7	Spring	Carbon steel
		Stainless steel high. Temp. Alloy
8	Adj. Screw	Bronze
		Carbon steel stainless steel
9	Adj. Screw lock nut	Bronze carbon steel
		Stainless steel
10	Spring guide	Carbon steel
		Stainless steel
11	Disc guide	Stainless steel
12	Blow down ring	Stainless steel
13	Lock bolt	Carbon steel
		Stainless steel
14	Gasket	Non-asbestos
		PTFE graphite
15	Stud. Bolts & nuts	Carbon steel
		Stainless steel
16	Lifting washer	Carbon steel
17	Lever	Ductile
18	Hex. Bolts	Carbon steel
		Stainless steel
19	Hex. Bolt & nut	Carbon steel
20	Hex. Bolt & nut	Carbon steel
21	Drain plug	Carbon steel
		Stainless steel
22	Fork lever	Ductile
23	Gasket	Non-asbestos
		PTFE graphite
24	Gasket	Non-asbestos
		PTFE graphite
25	O-ring	Viton
26	Wrench bolts	Stainless steel
27	Ball	Stainless steel
28	Bearing	Stainless steel

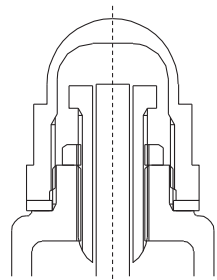
KSF Series Bellows Type
Size 1" (25A) ~ 8"(200A)



No.	Part Name	Standard Material
1	Body	Carbon steel
		Stainless steel
2	Bonnet	Carbon steel
		Stainless steel
3	Cap	Carbon steel
		Stainless steel
4	Seat	Stainless steel
5a	Disc	Stainless steel
5b	Disc holder	Stainless steel
6	Spindle	Carbon steel
		Stainless steel
7	Spring	Carbon steel
		Stainless steel high. Temp. Alloy
8	Adj. Screw	Carbon steel
		Stainless steel
9	Adj. Screw lock nut	Carbon steel stainless steel
		Carbon steel stainless steel
10	Spring guide	Carbon steel
		Stainless steel
11	Disc guide	Brass
		Stainless steel
12	Blow down ring	Brass
		Stainless steel
13	Lock bolt	Carbon steel
		Stainless steel
14	Gasket	Non-asbestos
		PTFE graphite
15	Stud. Bolts & nuts	Carbon steel
		Stainless steel
16	Drain plug	Brass
		Carbon steel stainless steel
17	Gasket	Non-asbestos
		PTFE graphite
18	Gasket	Non-asbestos
		PTFE graphite
19	Gasket	Viton
		Kalrez
20	Bellows	Stainless steel
21	Plate	Stainless steel
22	Middle guide	Carbon steel
		Stainless steel
23	Bearing	Stainless steel

Screwed Cap Type

Standard

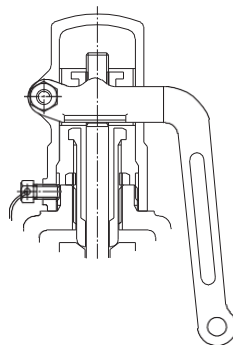


Screwed Cap and Test Gag Type

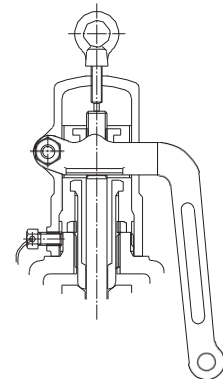
This test gag is handy for use in hydraulic static test. Replace with a plug and gasket after the test, otherwise the valve will not operate there by creating a dangerous situation.

Open Lever Type

This is used when hermetic sealing is not required. (with most uses for steam and air.)

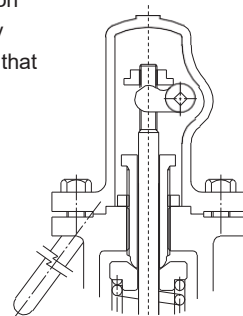


Open Lever and Test Gag Type

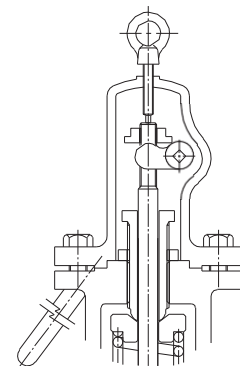


Packed Lever

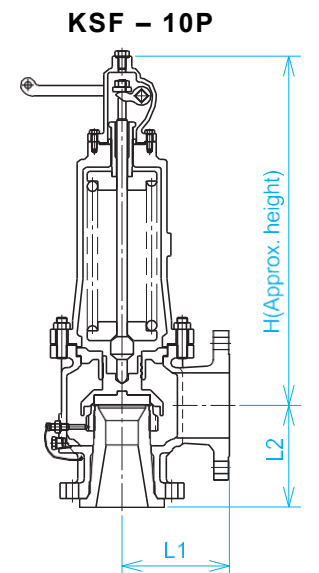
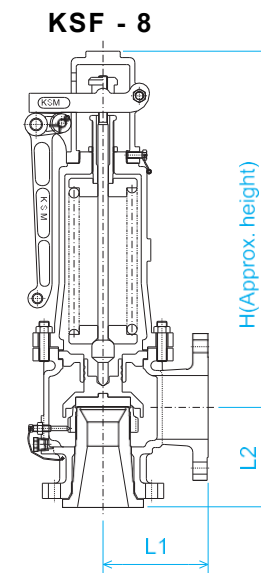
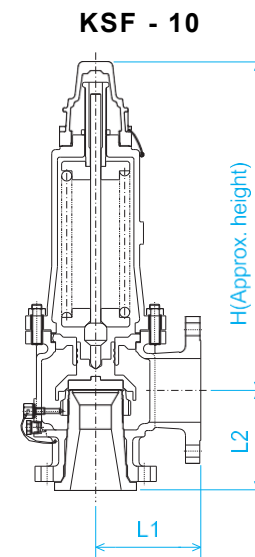
For services where tightness on the discharge side is necessary and where conditions are such that periodic testing is desirable.



Packed Lever and Test Gag



Application : Conventional Type



(Unit : mm)

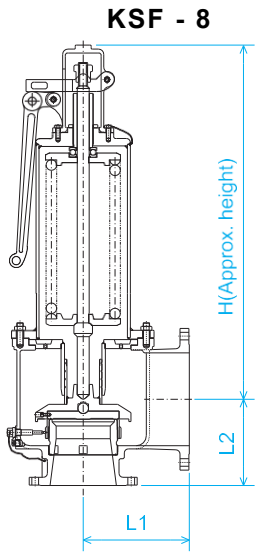
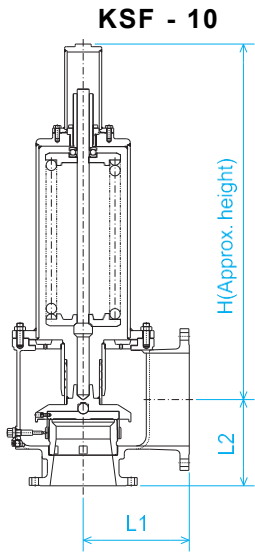
Size	Orifice	Center to face		Approx. Height			Approx. Weight
Inlet x Orifice Dia. x Outlet	Code	L1	L2		H		(kg)
				KSF-10	KSF-8	KSF-10P	
1/2" x 10 x 1"	D	96	92	194	196	254	6
3/4" x 10 x 1"	D	96	92	194	196	254	7
1" x 10 x 2"	D	114	105	228	246	280	12
1" x 13 x 2"	E	114	105	228	246	280	13
1" x 19" x 1.1/2"	*	114	105	228	246	280	13
1.1/4" x 24 x 2"	*	125	127	277	311	-	14
1.1/2" x 16 x 2"	F	121	124	304	333	364	20
1.1/2" x 21 x 2.1/2"	G	121	124	304	333	364	22
1.1/2" x 30 x 2.1/2"	*	121	124	304	333	364	22
1.1/2" x 26 x 3"	H	121	124	304	333	364	23
2" x 33 x 3"	J	124	137	340	369	400	28
2" x 38 x 3"	*	124	137	340	369	400	30
2.1/2" x 49 x 4"	L	169	153	490	562	538	57
3" x 39 x 4"	K	165	156	490	562	538	61
3" x 49 x 4"	L	165	156	490	562	538	56
3" x 61 x 4"	*	165	156	490	562	538	63
4" x 55 x 6"	M	184	178	625	668	738	85
4" x 60 x 6"	N	210	197	625	668	738	90
4" x 73 x 6"	P	210	197	625	668	738	100
5" x 95 x 8"	*	241	248	723	770	748	173
6" x 96 x 8"	Q	241	240	780	841	820	180
6" x 115 x 8"	R	241	240	780	841	820	200
8" x 150 x 10"	T	279	276	880	942	920	260

1) * : Special orifice area

2) Center to face tolerance : $\pm 1.5\text{mm}$ for valve inlet sizes up to and including 4"
 $\pm 3\text{mm}$ for valve inlet sizes larger than 4"

3) For inlet flange rating larger than 300 LBS may be change the dimension of center to face.

Application : Semi Nozzle Type



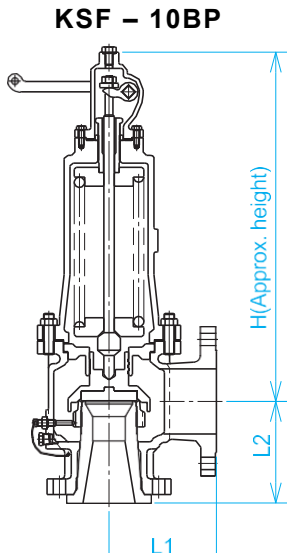
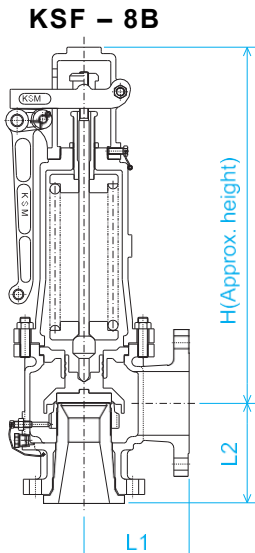
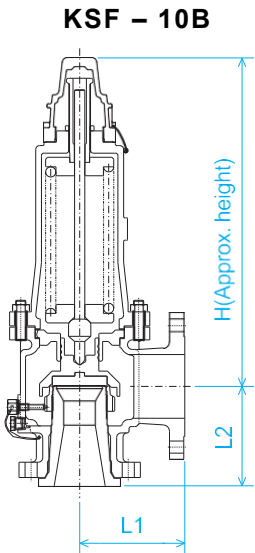
(Unit : mm)

Size	Orifice Code	Set Pressure Limit (kgf/ cm² g)	Center	To face	Approx. Height		Approx. Weight
Inlet x Orifice Dia. x Outlet			L1	L2	H		(kg)
					KSF-10	KSF-8	
10" x 204 x 14"	V	7	400	325	1340	1335	600
12" x 238 x 16"	W	7	400	375	1360	1355	650
14" x 278 x 18"	Y	7	500	400	1460	1455	750
16" x 292 x 18"	Z	7	500	425	1485	1480	850
18" x 357 x 24"	A	7	630	500	1765	1760	1500
20" x 397 x 24"	B	6	630	500	1935	1930	1800

1) Center to face tolerance : ±3mm

2) Inlet/outlet flange rating standard : ASME 150 LBS

Application : Bellows Type



(Unit : mm)

Size	Orifice Code	Center to Face		Approx. Height			Approx. Weight (kg)
Inlet x Orifice Dia. x Outlet		L1	L2	H			
				KSF-10B	KSF-8B	KSF-10BP	
3/4" x 10 x 1	D	96	92	226	228	287	10
1" x 10 x 2"	D	114	105	265	283	322	13
1" x 13 x 2"	E	114	105	265	283	322	14
1.1/2" x 16 x 2"	F	121	124	334	363	396	21
1.1/2" x 21 x 2.1/2"	G	121	124	334	363	396	24
1.1/2" x 26 x 3"	H	121	124	334	363	396	28
2" x 33 x 3"	J	124	137	393	422	455	35
3" x 39 x 4"	K	165	156	539	612	587	67
3" x 49 x 4"	L	165	156	539	612	587	63
4" x 55 x 6"	M	184	178	648	691	761	95
4" x 60 x 6"	N	210	197	648	691	761	100
4" x 73 x 6"	P	210	197	648	691	761	120
6" x 96 x 8"	Q	241	240	780	841	820	190
6" x 115 x 8"	R	241	240	780	841	820	210
8" x 150 x 10"	T	279	276	895	957	935	280

1) Center to face tolerance : ±1.5mm for valve inlet sizes up to and including 4"
±3mm for valve inlet sizes larger than 4"

2) For inlet flange rating lager than 300 LBS may be change the dimension of center to face.

STANDARD VALVE SELECTION TABLES

STANDARD VALVE SELECTION TABLES ASME I ,VIII, API RP520

Standard in the Firm

Materials			Type	Valve size	KSB6216 OR KSB1511 Flange Rating Raised Race	
Body & Bonnet	Seat & Disc	Spring		Inlet x orifice dia. x Outlet	Inlet	Outlet
Ductile	Stainless Steel	Carbon Steel	KSF-8			
Casting Steel			KSF-10			
Forged Steel				15A x10 x25A	10	10
				20A x 10 x 25A	10	10
				25A x 19 x40A	10	10
				40A x 30 x 65A	10	10
				50A x 38 x 80A	10	10
				65A x 49 x 100A	10	10
				80A x 61 x 100A	10	10
				100A x 73 x 150A	10	10
				125A x 95 x 200A	10	10
				150A x 115 x 200A	10	10
				200A x 150 x250A	10	10
Casting Steel	Stainless Steel	Carbon Steel	KSF-8			
Forged Steel			KSF-10			
				15A x10 x 25A	20, 30	10
				20A x10 x 25A	20, 30	10
				25A x19 x 40A	20, 30	10
				40A x30 x 65A	20, 30	10
				50A x 38 x 80A	20, 30	10
				65A x 49 x 100A	20, 30	10
				80A x 61 x 100A	20, 30	10
				100A x 73 x 150A	20, 30	10
				125A x 95 x 200A	20, 30	10
				150A x115 x 200A	20, 30	10
				200A x 150 x 250A	20, 30	10

(Orifice D Area) 0.11 in² 0.710cm²

0.785cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm ² g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃	to 232℃
						315℃	
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1" x D x 2"	150	150	20.0	13.0
			1" x D x 2"	300	150	40.0	20.0
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1" x D x 2"	150	150	13.0	5.6
			1" x D x 2"	300	150	40.0	20.0
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	3/4" x D x 1"	150	150	16.0	13.0
			3/4" x D x 1"	300	150	40.0	19.0
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	3/4" x D x 1"	150	150	13.0	5.6
			3/4" x D x 1"	300	150	19.0	16.0

(Orifice E Area) 0.196 in² 1.265cm²

1.32665cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm ² g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃	to 232℃
						315℃	
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1" x E x 2"	150	150	20.0	13.0
			1" x E x 2"	300	150	20.0	13.0
			1" x E x 2"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1" x E x 2"	150	150	13.0	5.6
			1" x E x 2"	300	150	20.0	20.0
			1" x E x 2"	600	150	40.0	

(Orifice F Area) 0.307 in² 1.981cm²

2.0096cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm ² g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃	to 232℃
						315℃	
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1. 1/2" x F x 2"	150	150	20.0	13.0
			1. 1/2" x F x 2"	300	150	20.0	13.0
			1. 1/2" x F x 2"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1. 1/2" x F x 2"	150	150	13.0	5.6
			1. 1/2" x F x 2"	300	150	20.0	20.0
			1. 1/2" x F x 2"	600	150	40.0	

STANDARD VALVE SELECTION TABLES ASME SEC I ,VIII, API RP520

(Orifice G Area) 0.503 in², 3.245cm²

3.46185cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm²g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃ to 232℃	315℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1.1/2" x G x 2.1/2"	150	150	20.0	13.0
			1.1/2" x G x 2.1/2"	300	150	20.0	20.0
			1.1/2" x G x 2.1/2"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1.1/2" x G x 2.1/2"	150	150	13.0	5.6
			1.1/2" x G x 2.1/2"	300	150	20.0	20.0
			1.1/2" x G x 2.1/2"	600	150	40.0	

(Orifice K Area) 1.838 in², 11.858cm²

11.93985cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm²g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃ to 232℃	315℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	3" x K x 4"	150	150	20.0	13.0
			3" x K x 4"	300	150	20.0	20.0
			3" x K x 4"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	3" x K x 4"	150	150	13.0	5.6
			3" x K x 4"	300	150	20.0	13.0
			3" x K x 4"	600	150	40.0	

(Orifice H Area) 0.785 in², 5.065cm²

5.3066cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm ² g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9 ℃ to 232 ℃	315 ℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1.1/2" x H x 3"	150	150	20.0	13.0
			1.1/2" x H x 3"	300	150	20.0	20.0
			2" x H x 3"	300	150	20.0	20.0
			2" x H x 3"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	1.1/2" x H x 3"	150	150	15.0	5.6
			1.1/2" x H x 3"	300	150	20.0	20.0
			2" x H x 3"	300	150	20.0	20.0
			2" x H x 3"	600	150	40.0	

(Orifice L Area) 2.853 in², 18.406cm²

18.84785cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm ² g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃ to 232℃	315℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	3" x L x 4"	150	150	20.0	13.0
			3" x L x 4"	300	150	20.0	20.0
			4" x L x 6"	300	150	20.0	20.0
			4" x L x 6"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	3" x L x 4"	150	150	13.0	5.6
			3" x L x 4"	300	150	20.0	13.0
			4" x L x 6"	300	150	20.0	13.0
			4" x L x 6"	600	150	40.0	

(Orifice J Area) 1.287 in², 8.303cm²

8.54865cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm ² g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃ to 232℃	315℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	2" x J x 3"	150	150	20.0	13.0
			2" x J x 3"	300	150	20.0	20.0
			2.1/2" x J x 4"	300	150	20.0	20.0
			2.1/2" x J x 4"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	2" x J x 3"	150	150	13.0	5.6
			2" x J x 3"	300	150	20.0	20.0
			2.1/2" x J x 4"	300	150	20.0	20.0
			2.1/2" x J x 4"	600	150	40.0	

(Orifice M Area) 3.60 in², 23.226cm²

11.93985cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm²g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃ to 232℃	315℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	4" x M x 6"	150	150	20.0	13.0
			4" x M x 6"	300	150	20.0	20.0
			4" x M x 6"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	4" x M x 6"	150	150	13.0	5.6
			4" x M x 6"	300	150	20.0	13.0
			4" x M x 6"	600	150	40.0	

(Orifice Q Area) 11.05 in², 71.29cm²

72.3456cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm²g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃ to 232℃	315℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	6" x Q x 8"	150	150	11.6	11.6
			6" x Q x 8"	300	150	20.0	20.0
			6" x Q x 8"	600	150	40.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	6" x Q x 8"	150	150	11.6	5.6
			6" x Q x 8"	300	150	20.0	11.6
			6" x Q x 8"	600	150	40.0	

(Orifice R Area) 16.0 in², 103.226cm²

103.81625cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm²g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃ to 232℃	315℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	6" x R x 8"	150	150	7.0	7.0
			6" x R x 8"	300	150	16.2	16.2
			6" x R x 8"	600	150	20.0	
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	6" x R x 8"	150	150	7.0	5.6
			6" x R x 8"	300	150	16.2	11.6
			6" x R x 8"	600	150	20.0	

(Orifice T Area) 26.0 in², 167.742cm²

176.625cm² KSM actual area

Materials		Type	Valve Size	ASME Flange Rating Raised Face		Maximum Press (kgf/cm ² g)	
Body & bonnet	Spring		Inlet x Orifice x Outlet	Inlet	Outlet	Inlet Temperature Range	
						-28.9℃ to 232℃	315℃
Carbon Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	8" x T x 10"	150	150	4.6	4.6
			8" x T x 10"	300	150	16.2	13.0
Stainless Steel	Carbon Steel Stainless Steel	KSF-8 KSF-10	8" x T x 10"	150	150	4.6	5.6
			8" x T x 10"	300	150	16.2	11.6

Code : KS B 6126 or KOSHA for Steam

W = 5.2456 x c x Kd x A x (P+0.1) x 0.9, See Table(page 32)

Nomenclature

W	Capacity of Steam, kg / h
A	Nozzle orifice area, mm ²
P	For Set Pressure ≤ 0.1 Mpag P = Set Pressure + 0.02 Mpag For Set Pressure > 0.1 > Mpag P = Set Pressure x 1.03
K d	Coefficient of discharge, 0.864(Full Lift Type)
C	Superheat Correction Factor See Table 2 (page 34)

Code : ASME Section I . For Steam

110kgf / cm² abs W = 51. 5 x K x A x P x 0.9 x Ksh

110kgf / cm² abs W = 51. 5 x K x A x P x 0.9 x Ksh x $\frac{(2.7109P - 1000)}{(3.2599P - 1061)}$

Code : ASME Section VIII. For Steam

For P 1 ≤ 117kgf / cm² abs W = 51. 5 x K d x A x P 1 x Ksh

For P 1 > 117kgf / cm² abs W = 51. 5 x K d x A x P 1 x Ksh $\frac{(2.7109P 1 - 1000)}{(3.2599P 1 - 1061)}$

Nomenclature

W	Weight of Steam (kg / h)
K	Coefficient of discharge = 0.975
K d	Coefficient of discharge = 0.878
A	Nozzle orifice area
P	(1.03 x Set Pressure) + 1.033 (kgf / cm ² abs)
P 1	(1.10 x Set Pressure) + 1.033 (kgf / cm ² abs)
K s h	Superheat Correction Factor See Table 3 (page 35,36)

SAFETY VALVE CAPACITY TABLE

Full Lift Type, For Set. Steam

Table 1

Size	15A	20A	25A			32A	40A			50A			65A	80A		100A	125A	150A	200A	
Orifice Dia. (mm)	10	10	10	13	19	24	16	21	30	26	33	38	49	49	61	73	95	115	150	
0.5	0.049	54.1	54.1	54.1	91.4	195	311	138	238	487	365	589	781	1299	1299	2013	2883	4884	7157	12176
0.7	0.069	60.5	60.5	60.5	102	218	348	154	266	544	409	659	873	1453	1453	2252	3225	5462	8004	13617
1	0.098	69.8	69.8	69.8	117	252	402	178	307	628	471	760	1008	1676	1676	2597	3720	6300	9232	15706
1.5	0.147	80.5	80.5	80.5	136	290	463	206	355	724	544	876	1162	1932	1932	2995	4290	7265	10647	18114
2	0.196	96.6	96.6	96.6	163	348	556	247	426	870	653	1052	1395	2321	2321	3597	5151	8724	12784	21750
2.5	0.245	112	112	112	190	407	649	288	497	1015	762	1228	1629	2709	2709	4198	6012	10182	14921	25386
3	0.294	128	128	128	217	465	742	330	568	1160	871	1404	1862	3097	3097	4799	6874	11641	17059	29023
3.5	0.343	145	145	145	245	524	836	371	640	1306	981	1580	2096	3485	3485	5401	7735	13100	19196	32659
4	0.392	161	161	161	272	582	929	412	711	1451	1090	1756	2329	3873	3873	6002	8596	14558	21334	36296
4.5	0.441	177	177	177	299	640	1022	454	782	1597	1199	1932	2562	4261	4261	6603	9457	16017	23471	39932
5	0.49	193	193	193	327	699	1115	495	853	1742	1309	2108	2796	4649	4649	7205	10319	17475	25608	43568
5.5	0.539	209	209	209	354	757	1208	537	925	1888	1418	2284	3029	5037	5037	7806	11180	18934	27746	47205
6	0.588	225	225	225	381	815	1301	578	996	2033	1527	2460	3262	5425	5425	8408	12041	20393	29883	50841
6.5	0.637	242	242	242	409	874	1394	619	1067	2179	1636	2636	3496	5813	5813	9009	12902	21851	32020	54478
7	0.686	258	258	258	436	932	1487	661	1139	2324	1746	2812	3729	6201	6201	9610	13764	23310	34158	58114
7.5	0.735	274	274	274	463	990	1580	702	1210	2470	1855	2988	3963	6589	6589	10212	14625	24768	36295	61750
8	0.784	290	290	290	491	1049	1673	743	1281	2615	1964	3164	4196	6977	6977	10813	15486	26227	38433	65387
8.5	0.833	306	306	306	518	1107	1767	785	1352	2760	2073	3340	4429	7365	7365	11414	16347	27686	40570	69023
9	0.882	322	322	322	545	1165	1860	826	1424	2906	2183	3516	4663	7753	7753	12016	17209	29144	42707	72659
9.5	0.931	339	339	339	573	1224	1953	868	1495	3051	2292	3692	4896	8141	8141	12617	18070	30603	44845	76296
9.9	0.97	351	351	351	594	1270	2027	901	1552	3167	2379	3832	5082	8450	8450	13096	18755	31764	46546	79190
10	0.98	355	355	355	600	1282	2046	909	1566	3197	2401	3868	5129	8529	8529	13219	18931	32061	46982	79932
10.3	1.009	364	364	364	616	1317	2101	933	1608	3283	2466	3972	5268	8759	8759	13575	19441	32925	48247	82084
10.5	1.029	371	371	371	627	1340	2139	950	1637	3342	2510	4044	5363	8917	8917	13820	19792	33520	49120	83569
11	1.078	387	387	387	655	1399	2232	992	1709	3488	2620	4220	5596	9305	9305	14421	20654	34979	51257	87205
11.5	1.127	403	403	403	682	1457	2325	1033	1780	3633	2729	4396	5830	9693	9693	15023	21515	36437	53394	90841
12	1.176	419	419	419	709	1515	2418	1074	1851	3779	2838	4572	6063	10081	10081	15624	22376	37896	55532	94478
12.5	1.225	436	436	436	736	1574	2511	1116	1923	3924	2947	4748	6296	10469	10469	16225	23237	39354	57669	98114
13	1.274	452	452	452	764	1632	2604	1157	1994	4070	3057	4924	6530	10857	10857	16827	24099	40813	59807	101751
13.5	1.323	468	468	468	791	1690	2697	1199	2065	4215	3166	5100	6763	11246	11246	17428	24960	42272	61944	105387
14	1.372	484	484	484	818	1749	2791	1240	2136	4360	3275	5276	6996	11634	11634	18030	25821	43730	64081	109023
14.5	1.421	500	500	500	846	1807	2884	1281	2208	4506	3384	5452	7230	12022	12022	18631	26682	45189	66219	112660
15	1.47	516	516	516	873	1865	2977	1323	2279	4651	3494	5628	7463	12410	12410	19232	27544	46647	68356	116296
15.5	1.519	533	533	533	900	1924	3070	1364	2350	4797	3603	5804	7697	12798	12798	19834	28405	48106	70493	119933
15.7	1.539	539	539	539	911	1948	3108	1381	2379	4856	3647	5876	7792	12956	12956	20079	28757	48701	71366	121417
16	1.568	549	549	549	928	1982	3163	1405	2421	4942	3712	5980	7930	13186	13186	20435	29266	49565	72631	123569
16.5	1.617	565	565	565	955	2040	3256	1447	2493	5088	3821	6156	8163	13574	13574	21037	30127	51023	74768	127205
17	1.666	581	581	581	982	2099	3349	1488	2564	5233	3931	6332	8397	13962	13962	21638	30989	52482	76906	130842
17.5	1.715	597	597	597	1010	2157	3442	1530	2635	5379	4040	6508	8630	14350	14350	22239	31850	53940	79043	134478
18	1.764	613	613	613	1037	2215	3535	1571	2707	5524	4149	6684	8863	14738	14738	22841	32711	55399	81180	138114
18.5	1.813	630	630	630	1064	2274	3628	1612	2778	5670	4258	6860	9097	15126	15126	23442	33573	56858	83318	141751
19	1.862	646	646	646	1092	2332	3721	1654	2849	5815	4368	7036	9330	15514	15514	24043	34434	58316	85455	145387
19.5	1.911	662	662	662	1119	2391	3815	1695	2920	5960	4477	7212	9564	15902	15902	24645	35295	59775	87593	149024
20	1.96	678	678	678	1146	2449	3908	1736	2992	6106	4586	7388	9797	16290	16290	25246	36156	61233	89730	152660
20.5	2.009	694	694	694	1173	2507	4001	1778	3063	6251	4695	7564	10030	16678	16678	25848	37018	62692	91867	156296
21	2.058	710	710	710	1201	2566	4094	1819	3134	6397	4805	7740	10264	17066	17066	26449	37879	64151	94005	159933
21.5	2.107	726	726	726	1228	2624	4187	1861	3205	6542	4914	7916	10497	17454	17454	27050	38740	65609	96142	163569
22	2.156	743	743	743	1255	2682	4280	1902	3277	6688	5023	8092	10730	17842	17842	27652	39601	67068	98279	167206
22.5	2.205	759	759	759	1283	2741	4373	1943	3348	6833	5132	8268	10964	18230	18230	28253	40463	68526	100417	170842
23	2.254	775	775	775	1310	2799	4466	1985	3419	6979	5242	8444	11197	18618	18618	28854	41324	69985	102554	174478
23.5	2.303	791	791	791	1337	2857	4559	2026	3491	7124	5351	8620	11431	19006	19006	29456	42185	71443	104692	178115
24	2.352	807	807	807	1365	2916	4652	2067	3562	7270	5460	8796	11664	19394						

SUPERHEAT CORRECTION FACTOR FOR SAFETY VALVES

Safety Valve Capacity Table for Steam (Table 2)

Pressure(absolute) Kgf cm²a(Mpa)	Saturation Temperature	Temperature (°C)																															
		200	220	240	260	280	300	320	340	360	380	400	420		440	460	480	500	520	540	560	580	600	620	640	660	680	700					
5(0.5)	1.005	0.996	0.972	0.951	0.931	0.913	0.896	0.879	0.864	0.849	0.835	0.822																					
10(1.0)	0.987	0.981	0.983	0.960	0.938	0.919	0.901	0.884	0.868	0.853	0.838	0.825																					
15(1.5)	0.977	0.967	0.970	0.972	0.947	0.925	0.906	0.888	0.872	0.856	0.841	0.828																					
20(2.0)	0.972		0.957	0.964	0.955	0.932	0.912	0.893	0.846	0.880	0.845	0.830	0.817		0.804	0.792	0.780	0.768															
25(2.5)	0.959			0.961	0.961	0.937	0.918	0.898	0.880	0.863	0.848	0.833	0.819		0.806	0.793	0.782	0.770															
30(3.0)	0.957			0.962	0.957	0.949	0.924	0.903	0.885	0.867	0.851	0.836	0.822		0.808	0.795	0.783	0.774	0.763	0.748	0.742	0.730	0.721	0.712	0.703	0.695	0.687	0.679					
40(4.0)	0.955				0.958	0.945	0.934	0.915	0.894	0.875	0.857	0.84	0.826		0.813	0.799	0.787	0.775	0.763	0.755	0.744	0.735	0.725	0.715	0.705	0.696	0.688	0.880					
50(5.0)	0.966					0.955	0.953	0.927	0.904	0.884	0.865	0.848	0.832		0.817	0.803	0.790	0.778	0.766	0.755	0.747	0.737	0.723	0.717	0.708	0.697	0.689	0.681					
60(6.0)	0.968					0.962	0.953	0.941	0.911	0.891	0.872	0.854	0.838		0.882	0.808	0.794	0.781	0.769	0.758	0.747	0.739	0.729	0.719	0.710	0.698	0.690	0.682					
70(7.0)	0.971						0.958	0.954	0.924	0.901	0.881	0.861	0.844		0.827	0.812	0.798	0.785	0.772	0.761	0.749	0.739	0.731	0.721	0.708	0.702	0.691	0.683					
80(8.0)	0.975						0.967	0.956	0.934	0.912	0.888	0.868	0.860		0.833	0.817	0.802	0.789	0.776	0.763	0.752	0.741	0.731	0.719	0.710	0.701	0.692	0.684					
90(9.0)	0.980							0.962	0.957	0.926	0.897	0.846	0.856		0.838	0.822	0.807	0.792	0.779	0.766	0.754	0.743	0.733	0.722	0.711	0.702	0.693	0.685					
100(10.0)	0.986							0.971	0.961	0.936	0.909	0.883	0.883		0.844	0.827	0.811	0.796	0.782	0.776	0.757	0.745	0.735	0.724	0.712	0.703	0.695	0.686					
120(12.0)	0.999								0.975	0.964	0.926	0.903	0.876		0.857	0.838	0.818	0.805	0.789	0.775	0.768	0.750	0.739	0.728	0.718	0.706	0.697	0.688					
140(14.0)	1.016									1.002	0.980	0.936	0.893		0.868	0.846	0.828	0.811	0.797	0.782	7.768	0.755	0.743	0.732	0.721	0.711	0.699	0.691					
160(16.0)	1.036										1.000	0.964	0.907		0.883	0.858	0.838	0.819	0.803	0.787	0.774	0.760	0.748	0.736	0.725	0.714	0.704	0.693					
180(18.0)	1.063										1.038	0.980	0.929		0.895	0.873	0.848	0.828	0.810	0.794	0.779	0.766	0.752	0.740	0.728	0.717	0.707	0.697					
200(20.0)	1.094											1.000	1.006	0.963		0.914	0.885	0.861	0.835	0.818	0.801	0.786	0.770	0.757	0.744	0.732	0.720	0.710	0.700				
220(22.0)	1.129											1.038	1.033	0.982		0.932	0.900	0.872	0.849	0.827	0.808	0.793	0.777	0.761	0.749	0.736	0.724	0.713	0.702				
240(24.0)													1.059	1.016		0.968	0.915	0.885	0.861	0.837	0.815	0.797	0.83	0.766	0.752	0.740	0.727	0.716	0.705				
260(26.0)													1.099	1.055		0.982	0.935	0.899	0.871	0.848	0.825	0.804	0.786	0.722	0.756	0.74	0.731	0.719	0.708				
280(28.0)													1.167	1.096		1.013	0.956	0.913	0.833	0.853	0.834	0.811	0.793	0.776	0.762	0.747	0.735	0.720	0.710				
300(30.0)														1.132		1.047	0.977	0.931	0.895	0.867	0.838	0.821	0.799	0.781	0.763	0.753	0.735	0.724	0.715				
320(32.0)													1.169			1.089	1.009	0.952	0.908	0.877	0.849	0.824	0.805	0.787	0.770	0.735	0.742	0.729	0.174				
340(34.0)																1.136	1.032	0.968	0.823	0.888	0.859	0.835	0.812	0.792	0.775	0.757	0.746	0.729	0.718				
360(36.0)																1.191	1.063	0.989	0.941	0.899	0.889	0.842	0.818	0.798	0.780	0.761	0.750	0.734	0.723				
380(38.0)																	1.098	1.016	0.956	0.913	0.878	0.850	0.823	0.804	0.785	0.765	0.750	0.739	0.726				
400(40.0)																		1.137	1.037	0.972	0.927	0.888	0.858	0.832	0.807	0.790	0.789	0.754	0.742	0.725			
420(42.0)																				1.064	0.995	0.944	0.901	0.868	0.839	0.815	0.792	0.774	0.758	0.745	0.729		
440(44.0)																					1.092	1.012	0.954	0.914	0.876	0.846	0.821	0.800	0.779	0.762	0.748	0.731	
460(46.0)																						1.122	1.035	0.971	0.924	0.888	0.854	0.828	0.805	0.785	0.766	0.753	0.738

Pressure = Set Pressure + Over Pressure + Atmospheric Pressure 1 Psig = 0.06895 barg = 0.070.1 kgf / cm²g 1barg = 14.5035 psig = 1.0197 kgf / kgf / cm²g

27 www.gvk.co.kr

Pressure = Set Pressure + Over Pressure + Atmospheric Pressure 1 Psig = 0.06895 barg = 0.070.1 kgf / cm²g 1barg = 14.5035 psig = 1.0197 kgf / kgf / cm²g

CALCULATION OF SAFETY VALVE CAPACITY

For Gas or Vapor (HPGCL)

W = (C x Kd x Kb x Pc x A x sqrt(M)) / (13160 x sqrt(T x Z))

W	Capacity of Gas or Vapor (kg / h)	A	Orifice area, mm²
C	Coefficient Determined Form an expression of the ratio of the specific heats (k = C p / C v) of the Gas or Vapor at inlet relieving conditions. (See Table 4)	P 1	Upstream relieving pressure absolute, kPag
K d	Coefficient of discharge, Safety valve 0.975	M	Molecular weight of the gas or vapor at inlet relieving conditions
K b	Capacity correction factor due to back pressure. For conventional type valve, use a valve for K b equal to 1.0	Z	Compressibility factor for the deviation of the actual gas from a perfect gas, a ratio evaluated at inlet Relieving conditions.(if a calculated compressibility is not Available, a Z value of 1.0 should be used.)
C	Capacity correction factor for installations with a rupture Disk Upstream of the safety valve 1.0 When a rupture disk is not installed.	T	Relieving temperature of the inlet gas or vapor, K (°C + 273)

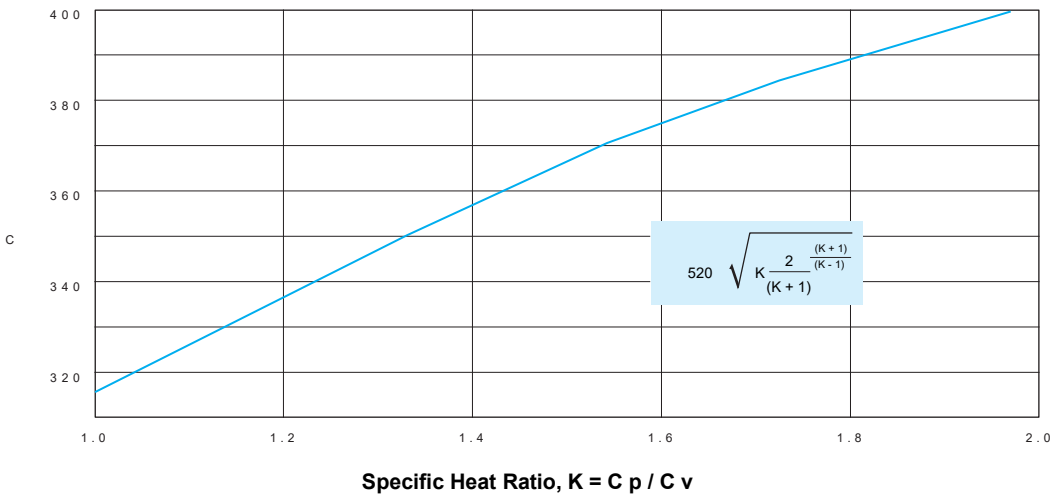


Table 4

k	C	k	C	k	C	k	C
1	315	1.2	337	1.4	356	1.6	373
1.02	318	1.22	339	1.42	358	1.62	374
1.04	320	1.24	341	1.44	360	1.65	376
1.06	322	1.26	343	1.46	361	1.66	377
1.08	325	1.28	345	1.48	363	1.68	379
1.1	327	1.3	347	1.5	365	1.7	380
1.12	329	1.32	349	1.52	366	1.8	387
1.14	331	1.34	351	1.54	368	1.9	394
1.16	333	1.36	353	1.56	369	1.95	397
1.18	335	1.38	354	1.58	371	2	400

For Water (HPGCL)

W = (Kd x Kw x Kc x Kv x Kp x A x (1.25p - Pb)^(1/2)) / (11.78 x (G)^(1/2))

A	Effective discharge area, mm²	G	Specific gravity of the liquid at the flowing temperature referred to water at standard conditions.
W	Flow rate, L/min	P	Set pressure, kPag
K d	Rated coefficient of discharge that should be obtained from the valve manufacturer. Safety valve 0.65.	P b	Total back pressure, kPag
K W	Correction factor due to back pressure. If back pressure is atmospheric, Kw=1. Balanced bellows valves in back pressure service will require the correction factor determined from Figure 5. Conventional valves require no special correction.	K v	Correction factor due to viscosity as determined from Figure 6 or from the following equation: = (0.9935 + (2.878 / R^0.5) + (342.75 / R^0.5))^-1.0 R = (85,200 / (U * sqrt(A)))
K c	Combination correction factor for installation with a rupture disk upstream of the safety valve. Use a value for Kc equal to 1.0 when a rupture disk does not exist.	R	Reynolds Number
K p	Correction factor due to overpressure. At 25% overpressure, Kp=1.0. For overpressures other than 25%, Kp is determined from Figure 7	u	Viscosity at the flowing temperature, in Saybolt Universal seconds, SSU.

Figure 5
Correction Factor, Kw Due to back Pressure on Balanced Bellows

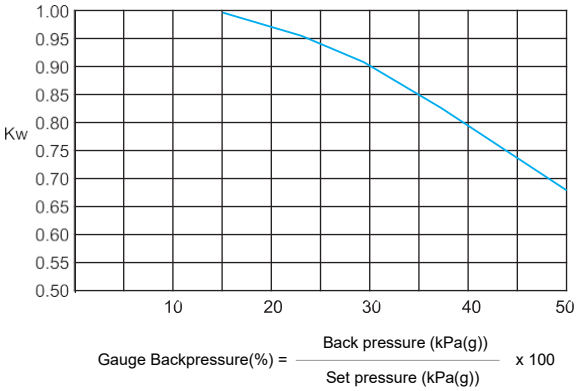


Figure 7
Correction Factor, Kp Due to Overpressure

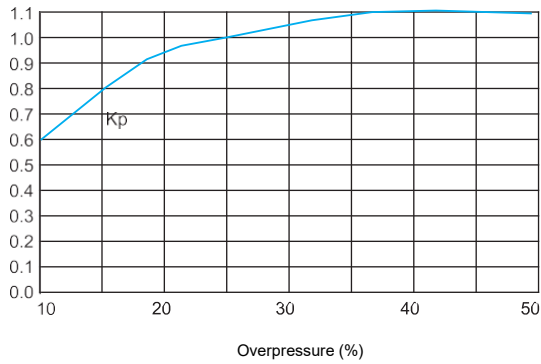
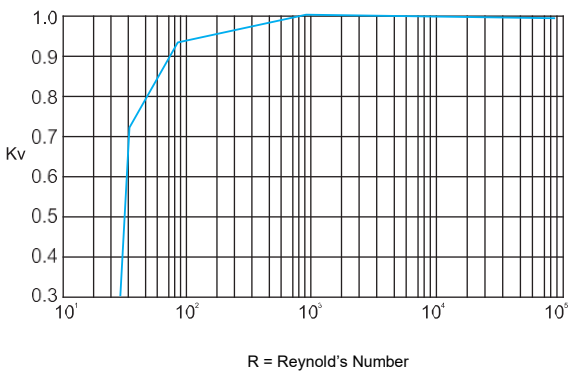


Figure 6
Correction factor, Kv, Due to Viscosity



Specific Heat Ratios Are At
Atmospheric Pressure and 60 °F Unless Otherwise Noted

Table 5

Chemical medium	Gas & Vapor Phase			Liquid Phase				
	K	M	SP.GR (AIR=1)	WATER=1 SP.GR	SP.GR@ TEMP °F	B.P.	Critical Temp °F	Critical pressure psia
	Cp/Cv							
Acetaldehyde	1.14	44	1.519	783	64	68	370	
Acetic acid	1.15	60	2.071	1.049	68	245	611	
Acetone	-	-	-	791	66	133	455	
Acetylene	1.26	26	0.898	-	-	-119	97	911
Air	1.4	29	1	-	-	-313	-220	547
Ammonia	1.31	17	0.587	817	-110	-28	270	1683
Argon	1.67	39.94	1.381	1.65	-387	-301	-188	
Benzene	1.12	78	2.7	0.879	68	176	551	700
Butadiene 1.3	1.12	54	1.922	0.621	68	24	306	661
Butane n	1.094	58	2.07	0.579	68	31	306	551
Iso	1.094	58	2.07	0.557	68	11	273	529
Carbon dioxide	1.29	44	1.53	1.101	-35	-109	88	1072
Carbon disulfide	1.21	76	2.628	1.263	68	116	523	
Carbone monoxide	1.4	28	967	0.814	-318	-314	-218	
Chlorine	1.36	70.9	2.45	1.56	-29	-30	291	
Cyclohexane	1.09	84	2.905	0.779	68	177	538	593
N-decane	1.03	142	4.91	0.734	60	345	655	320
Dowtherm a	1.046	166	5.696	0.997	212	496	-	
Dowtherm b	-	147	5.074	1.181	212	352	-	
Ethane	1.19	30	1.05	0.546	-126	-128	90	708
Ethyl alcohol	1.13	46	1.59	0.789	68	173	469	
Ethyl chloride	1.19	64.5	2.22	0.903	50	54	369	
Ethylene(ethene)	1.26	28.05	0.997	0.566	-152	-155	49	749
Freon 11	1.14	137.3	4.742	1.494	63	75	386	
Freon 12	1.14	120.9	4.174	1.486	-22	-21	234	
Freon 22	1.18	86.47	2.985	1.419	-42	-41	205	
Freon 114	1.09	170.92	5.9	1.538	30	38	295	
Gasoline	-	-	-	7.5**	68	158***	-	
Helium	1.66	4	0.138	-	-	-452	-450	
N-hexane	1.06	86	2.97	0.659	68	156	454	440
Hydrogen chloride	1.41	36.4	1.27	-	-	-118	127	
Hydrogen	1.41	2	0.07	0.709	-423	-423	-400	188
Hydrogen sulfide	1.32	34	1.19			-75	213	1306
Kerosene	-	-	-	0.815	60	-	-	
Methane	1.31	16	555	0.415	-263	-259	-116	673
Methyl alcohol	1.2	32	1.11	0.792	68	149	464	
Methyl butane	1.08	72.15	2.49	0.625	60	82	370	476
Methyl chloride	1.2	50.49	1.742	0.952	32	-11	290	
Natural gas	1.27	19**	0.656	-	-	-	-	
Nitric acid(hno2)	-	-	-	1.502	60	187	-	
Nitric oxide	1.4	30	1.036	1.269	-239	-240	-137	
Nitrogen	1.4	28	0.967	1.026	-422	-321	-233	
Nitrous oxide	1.3	44	1.519	1.226	-128	-131	98	
Nonane	1.04	128	4.43	0.718	68	303	613	345
N-octane	1.05	114	3.94	0.707	68	258	565	362
Oils.								
Fuel, bunker c	-	-	-	1.014	60	-	-	
Fuel, no.3	-	-	-	0.899	60	-	-	
Fuel, no.5 & 6	-	-	-	0.993	60	-	-	
Lube	-	-	-	0.91	60	-	-	
Mineral	-	-	-	0.91	60	-	-	
Oxygen	1.4	32	1.1	1.426	-422	-297	-182	
N-pentane	1.07	72	2.49	0.631	60	97	386	490
Propane	1.13	44	1.55	0.585	-49	-44	206	617
Propylene	1.15	42	1.476	0.609	-53	-54	197	667
Steam	1.33	18	0.622	1.00	39	212	706	3206
Styrene	1.07	104	3.6	0.906	68	293	706	
Sulfur dioxide	1.29	64	2.26	1.434	32	14	315	1141
Sulfuric acid	-	-	-	1.834	60	644***	-	
Toluene	1.09	92	3.18	0.866	68	231	609	611
N-heptane	1.05	100	3.49	0.684	60	209	513	397
Phenol	1.3	94	3.27			358		890

Constant back pressure only - Orifice area calculations

USA Unit formulas

Metric Unit Formulas

Gases or Vapors

Gases or Vapors (Lbs / hr)

$$A = \frac{W \sqrt{T Z}}{C K_d P_1 \sqrt{M K_b}}$$

Gases or Vapors (S.C.F.M)

$$A = \frac{V \sqrt{T Z M}}{6.32 C K_d P_1 K_b}$$

- or -

$$A = \frac{V \sqrt{T Z G}}{1.175 C K_d P_1 K_b}$$

Air... (S.C.F.M)

$$A = \frac{V_a \sqrt{T}}{418 K_d P_1 K_b}$$

Steam... (Lbs / hr)

$$A = \frac{W_s}{51.5 K_d P_1 K_b K_{sh}}$$

Liquid ... (G. P. M)

$$A = \frac{V_{g p m} \sqrt{G}}{38.0 K_d K_p \sqrt{1.25 P - P_b K_v}}$$

Variable or Constant back pressure

The Balanced Bellows valves must be used when the variation in back pressure exceeds 10% of the set pressure. Sizing formulas of Balanced Bellows valves for gas or vapor are the same as above formulas of Conventional valves except that the factor Kb changes to Kb.

The formulas of Liquid are as follows :

G. P. M

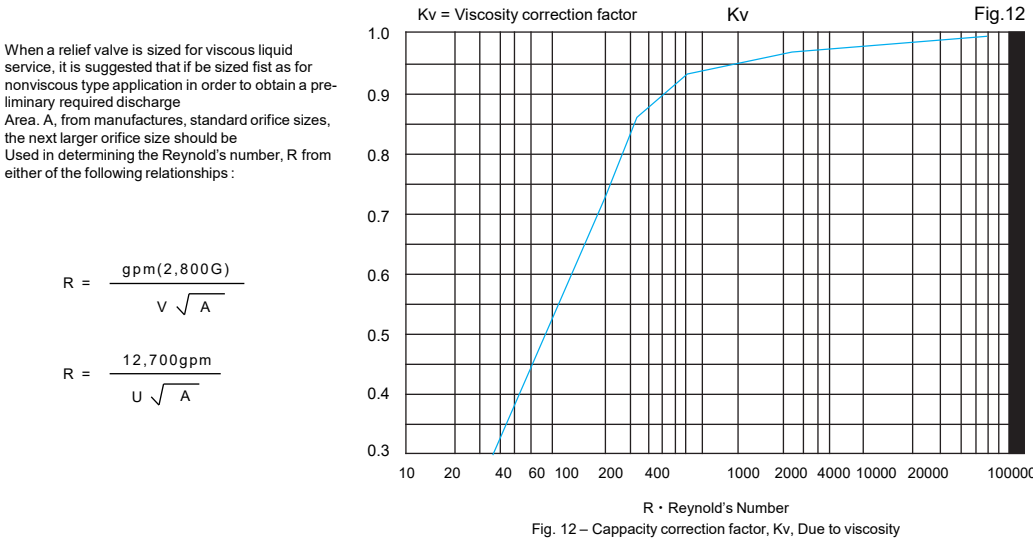
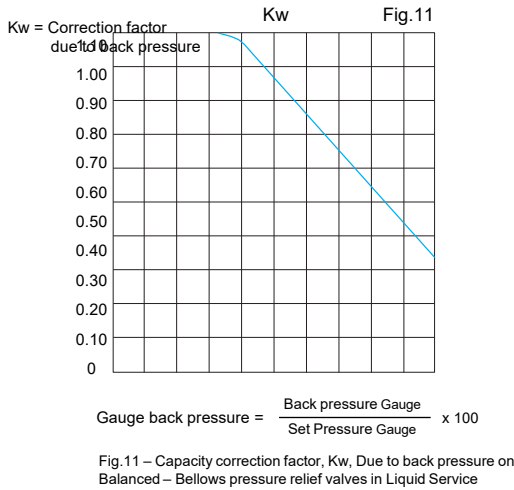
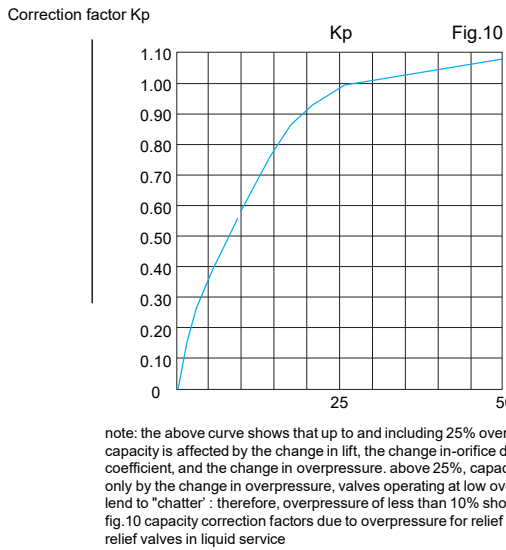
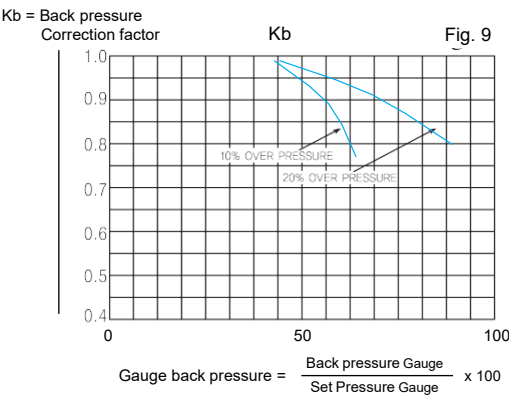
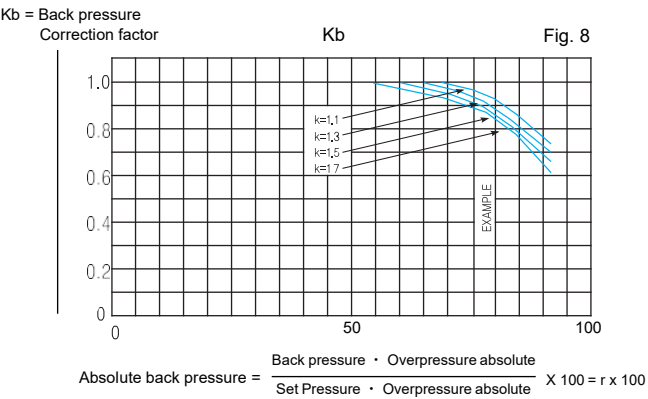
$$A = \frac{V_{g p m} \sqrt{G}}{38.0 K_d K_p \sqrt{1.25 P - P_b K_v}}$$

Liquids

liter / min

$$A = \frac{V_{\ell} \sqrt{G}}{84.0 K_d K_p \sqrt{1.25 P - P_b K_v}}$$

(Unit : mm)		
Nomenclature	USA UNIT Formulas	Metric UNIT Formulas
A = Required orifice area	in ²	cm ²
W = Required flow through valve	Lbs/hr	kg/hr
C = Coefficient determined by ratio of the specific heats of the gas of vapor.		
$C = 520 \sqrt{k \left(\frac{2}{K + 1} \right)^{\frac{k+1}{k-1}}}$ (in case of Lbs, in Runkin UNIT)		$C = 520 \sqrt{k \left(\frac{2}{K + 1} \right)^{\frac{k+1}{k-1}}}$ (in case of kg, cm Kelvin UNIT)
of select from table 6 and 7 on page 43 K = ratio of the specific heats, Cp/Cv of the gas or vapor This value is constant for an ideal gas. if this ratio is unknown, the value k=1.001 may be used.		
Kd = Coefficient of discharge		
= 0.975 for gas, vapor, air, steam for API sizing = 0.62 for liquids method = 0.869 for gas, vapor, air steam for ASME 90% of = 0.558 for liquids actual capacity		
V = Required flow through valve at 14.7 psia and 60 °F (in USA unit) at 1.0332kgf/cm ² abs and 0 °C (in Metric unit)	S.C.F.M. Psia (set P+over P+14.7)	NM3/hr. kgf/cm2abs (set P-over P+1.033)
P1 = Upstream relieving pressure absolute This is the set pressure plus the allowable over pressure, plus the atmospheric pressure.		
Kb = Constant back pressure correction factor		
$= \frac{735}{C} \sqrt{\frac{k}{k-1} \left(\frac{P2}{P1} \right)^{\frac{2}{k}} - \left(\frac{P2}{P1} \right)^{\frac{k}{k-1}}}$ (USA UNIT)	psia	
$= \frac{548}{C} \sqrt{\frac{k}{k-1} \left(\frac{P2}{P1} \right)^{\frac{2}{k}} - \left(\frac{P2}{P1} \right)^{\frac{k}{k-1}}}$ (Metric UNIT)		kgf/cm ² abs.
or from Fig 8 on page 42 P1 and P2=Upstream pressure and back pressure in absolute. kb=1.0 when back pressure is below 50% of abs. relieving pressure.		
M = Molecular weight of the gas or vapor.		
T = Absolute temperature of the inlet.	°F + 460	°C -273
Z = Compressibility factor corresponding to P1 and T. (if this factor is not available, compressibility correction can be safely ignored by using a value of Z=1.0)		
G = Specific gravity of gas(air=1.0) or specific gravity of liquid(water=1.0)		
Va = Required air flow through valve.	S.C.F.M.	NM3/hr kgf/hr
Ws = Required steam flow through valve		
Ksh = 1.0 for sat. steam Vgpm=Required liquid flow / Ksh = Super heat correction factor	Lbs/hr	
V ℓ = Required liquid flow	U.S	
Kp = Liquid capacity correction factor due to overpressure. Kp = 1.00 at 25% overpressure The factor for other overpressure can be obtained from Fig 10 on page 42	gallons/mi	
Kv = Viscosity correction factor from curve Fig 12 on page 42 Kv = 1.0 at normal viscosity		Liter / min
Kw = Back pressure correction factor for liquid service from Fig 11 on page 42 Balanced bellows type only.		
Kb = Back pressure correction factor for gas or vapor service from Fig 9 on page 42 Balanced bellows type only.		
P = Set pressure at inlet		kgf/cm ² g
Pb = Back pressure at outlet.	psig	kgf/cm ² g



Where
gpm = flow rate at the flowing temperature, in U.S gallons per minute.
G = specific gravity of the liquid at the flowing temperature referred to water = 1.00 at 70 °F.
V = absolute viscosity at the flowing temperature, in Saybolt universal seconds. After the valve of R is determined, the factor Kv is obtained from Fig.12. Fac-corrected area exceeds the "chosen standard orifice area" if the above calculations should be repeated using the next larger standard orifice (Per API RP520).

Values of Coefficient C(Per API RP 520)

In USA Unit Table 6

kn	C	kn	C	kn	C	kn	C	kn	C	kn	C
0.41	219.28	0.71	276.09	1.01	316.56	1.31	347.91	1.61	373.32	1.91	394.56
0.42	221.59	0.72	277.64	1.02	317.74	1.32	348.84	1.62	374.09	1.92	395.21
0.43	223.86	0.73	279.18	1.03	318.9	1.33	349.77	1.63	374.85	1.93	395.86
0.44	226.1	0.74	280.7	1.04	320.05	1.34	350.68	1.64	375.61	1.94	396.5
0.45	228.3	0.75	282.2	1.05	321.19	1.35	351.6	1.65	376.37	1.95	397.14
0.46	230.47	0.76	283.69	1.06	322.32	1.36	352.5	1.66	377.12	1.96	397.78
0.47	232.61	0.77	285.16	1.07	323.44	1.37	353.4	1.67	377.86	1.97	398.41
0.48	234.71	0.78	286.62	1.08	324.55	1.38	354.29	1.68	378.61	1.98	399.05
0.49	236.78	0.79	288.07	1.09	325.65	1.39	355.18	1.69	379.34	1.99	399.67
0.5	238.83	0.8	289.49	1.1	326.75	1.4	356.06	1.7	380.08	2	400.3
0.51	240.84	0.81	290.91	1.11	327.83	1.41	356.94	1.71	380.8	2.01	400.92
0.52	242.82	0.82	292.31	1.12	328.91	1.42	357.81	1.72	381.53	2.02	401.53
0.53	244.78	0.83	293.7	1.13	329.98	1.43	358.67	1.73	382.52	2.03	402.15
0.54	246.72	0.84	295.07	1.14	331.04	1.44	359.53	1.74	382.97	2.04	402.76
0.55	248.66	0.85	296.43	1.15	332.09	1.45	360.88	1.75	383.68	2.05	403.37
0.56	250.5	0.86	297.78	1.16	333.14	1.46	361.23	1.76	384.39	2.06	403.97
0.57	252.36	0.87	299.11	1.17	334.17	1.47	362.07	1.77	385.09	2.07	404.58
0.58	254.19	0.88	300.43	1.18	335.2	1.48	362.9	1.78	385.79	2.08	405.18
0.59	256	0.89	301.73	1.19	336.22	1.49	363.74	1.79	386.49	2.09	405.77
0.6	257.79	0.9	303.04	1.2	337.24	1.5	364.56	1.8	387.18	2.1	406.37
0.61	259.55	0.91	304.33	1.21	338.24	1.51	365.39	1.81	387.87	2.11	406.96
0.62	261.29	0.92	305.6	1.22	339.24	1.52	366.2	1.82	388.56	2.12	407.55
0.63	263.01	0.93	306.86	1.23	340.23	1.53	367.01	1.83	389.24	2.13	408.13
0.64	264.71	0.94	308.11	1.24	341.22	1.54	367.82	1.84	389.92	2.14	408.71
0.65	266.4	0.95	309.35	1.25	342.19	1.55	368.62	1.85	390.59	2.15	409.29
0.66	268.06	0.96	310.58	1.26	343.16	1.56	369.41	1.86	391.26	2.16	409.87
0.67	269.7	0.97	311.8	1.27	344.13	1.57	370.21	1.87	391.93	2.17	410.44
0.68	271.33	0.98	313.01	1.28	345.08	1.58	370.99	1.88	392.59	2.18	411.01
0.69	272.93	0.99	314.19	1.29	346.03	1.59	371.77	1.89	393.25	2.19	411.58
0.7	274.52	1	315.38	1.3	346.98	1.6	372.55	1.9	393.91	2.2	412.15

Interpolated values C becomes indeterminate as either n or K approachers 1.00.

k : Ratio of specific heat. $K = \frac{C_p}{C_v}$ n : isentropic expansion coefficient.

C : Coefficient determined by the ratio of the specific heats of the gas or vapor at standard condition

Values of Coefficient C

In Metric Unit Table 7

kn	C	kn	C	kn	C	kn	C	kn	C	kn	C
0.41	163.44	0.71	205.79	1.01	235.95	1.31	259.32	1.61	278.26	1.91	294.09
0.42	165.16	0.72	206.94	1.02	236.83	1.32	260.01	1.62	278.83	1.92	294.57
0.43	166.86	0.73	208.09	1.03	237.69	1.33	260.7	1.63	279.4	1.93	295.06
0.44	168.52	0.74	209.22	1.04	238.55	1.34	261.38	1.64	279.96	1.94	295.53
0.45	170.16	0.75	210.34	1.05	239.4	1.35	262.07	1.65	280.53	1.95	296.01
0.46	171.78	0.76	211.45	1.06	240.24	1.36	262.74	1.66	281.09	1.96	296.49
0.47	173.38	0.77	212.55	1.07	214.08	1.37	263.41	1.67	281.64	1.97	296.96
0.48	174.94	0.78	213.63	1.08	241.91	1.38	264.07	1.68	282.2	1.98	297.43
0.49	176.49	0.79	214.71	1.09	242.73	1.39	264.74	1.69	282.74	1.99	297.9
0.5	178.01	0.8	215.77	1.1	243.55	1.4	265.39	1.7	283.29	2	298.37
0.51	179.51	0.81	216.83	1.11	244.35	1.41	266.05	1.71	283.83	2.01	298.83
0.52	180.99	0.82	217.88	1.12	245.16	1.42	266.7	1.72	284.38	2.02	299.28
0.53	182.45	0.83	218.91	1.13	246.95	1.43	267.34	1.73	284.91	2.03	299.74
0.54	183.89	0.84	219.93	1.14	246.74	1.44	267.98	1.74	285.45	2.04	300.2
0.55	185.34	0.85	220.95	1.15	247.53	1.45	268.61	1.75	285.98	2.05	300.62
0.56	186.71	0.86	221.95	1.16	248.31	1.46	269.24	1.76	286.51	2.06	301.1
0.57	188.1	0.87	222.94	1.17	249.08	1.47	269.67	1.77	287.03	2.07	301.56
0.58	189.46	0.88	223.93	1.18	249.84	1.48	270.5	1.78	287.55	2.08	302
0.59	190.81	0.89	224.9	1.19	250.6	1.49	271.12	1.79	288.07	2.09	30.44
0.6	192.15	0.9	225.87	1.2	251.36	1.5	271.73	1.8	288.59	2.1	302.89
0.61	193.46	0.91	226.83	1.21	252.11	1.51	272.35	1.81	289.1	2.11	303.33
0.62	194.75	0.92	227.78	1.22	252.85	1.52	272.95	1.82	289.62	2.12	303.77
0.63	196.04	0.93	228.72	1.23	253.59	1.53	273.55	1.83	290.12	2.13	304.2
0.64	197.31	0.94	229.65	1.24	254.33	1.54	274.16	1.84	290.63	2.14	304.63
0.65	198.56	0.95	230.58	1.25	255.05	1.55	274.75	1.85	291.13	2.15	305.07
0.66	199.8	0.96	231.49	1.26	255.78	1.56	275.34	1.86	291.63	2.16	305.5
0.67	201.02	0.97	232.4	1.27	256.5	1.57	275.94	1.87	292.13	2.17	305.92
0.68	202.24	0.98	233.3	1.28	257.21	1.58	276.52	1.88	292.62	2.18	306.35
0.69	203.43	0.99	234.18	1.29	257.92	1.59	277.1	1.89	293.11	2.19	306.77
0.7	204.62	1	235.07	1.3	258.62	1.6	277.68	1.9	293.6	2.2	307.2

Interpolated values since C becomes indeterminate as either n or K approachers 1.00.

k : Ratio of specific heat. $K = \frac{C_p}{C_v}$ n : isentropic expansion coefficient.

C : Coefficient determined by the ratio of the specific heats of the gas or vapor at standard condition

1. Outline

It is necessary to select the type, size, material, etc of the safety valve matching to the facility to prevent disasters such as damage or explosion of utensil resulted from the extraordinarily increasing internal pressure by installing it in the pressure-generating device such as gas, heat, etc or a pressure vessel to maintain it. If it is not properly selected, sometimes it does not function sufficiently as a safety valve to cause a disaster. Therefore, we will indicate the note for handing that you should know as a final user(consumer) to exactly use the safety valve and explain its basic points.

2. Definition and type of a safety valve

2-1. Definition of a safety valve

A safety valve should keep the original airtight condition by discharging a fluid of excessive pressure automatically not depending on other assistant power or human power and reducing the pressure in the equipment to the fixed one. The safety valve is defined as follows in ISO international standard. A safety valve is the valve to be closed by automatically emitting the pressure by a fluid without any other assistance except for its own power and recovering it to the normal when the pressure of a fluid exceeds the fixed pressure. Also, in Japanese Industrial Standard, it is explained as a valve with the ability to emit the fluid(steam and gas) of nominal emitting amount by automatically operating at a pre-fixed pressure at the entrance and becoming a normal condition again when the pressure decreases.

2-2 Type of a safety valve

There are various types of different structures but a lot of them cannot be found in the existing market so we mention the general spring-type safety valve here.

a. Lift and full bore safety valve

A safety valve can be classified into lift safety valve and full bore safety valve according to the type of a device.(KS B 6216-1998)

- Lift safety valve (Figure 13)

The lift shall be more than 1/40 of the seat caliber or be less than 1/4 and with the disc opened, the area of the fluid passage of the seat caliber shall be minimum.

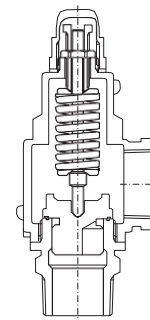


Figure 13
Lift Safety Valve

- Full bore safety valve (Figure 14)

The seat caliber shall exceed 1.15 times of that of the neck. The area of the fluid passage of the seat orifice when the disc is opened shall exceed 1.05 times of neck caliber while the area of the fluid passage of safety valve entrance and pipe shall exceed 1.7 times.
(Refer to the catalogue for the names of each part.)

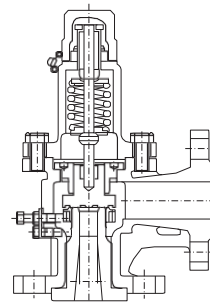


Figure 14
Full Bore Safety Valve

B. Types according to the differences of forms

- Lever type (Figure 15)

A lever is to check any defect of lift device on the pressurized condition and it is installed for the manual discharge from the safety valve irrespective of the operation condition to promptly discharge with parts of heat on operation. Provided that it is inappropriate for toxic gas, heating gas, liquid, etc.

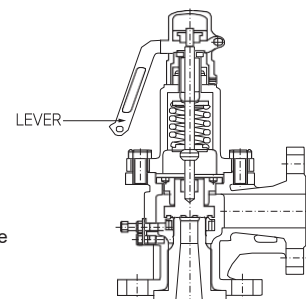


Figure 15
Lever Type

- Closed Type

The structure of a valve is closed as it is dangerous to discharge toxic gas or heating gas close to the facility.

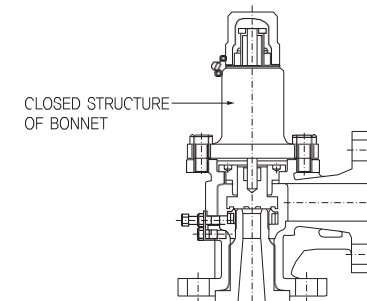


Figure 16
Closed Type

- Top Guide

The disc is of piston structure for the smooth upward and downward movement of disc, which is supported by valve guide of cylinder type. The both of entrance and exit have less pressure loss and good discharge coefficient.

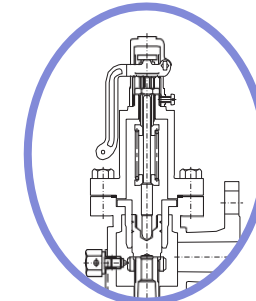


Figure 18
Top Guide

- Soft Seat

It uses the soft synthetic resin on the seat of the safety valve and thanks to the elasticity, it attaches to the seat well. Provided that the application scope of resin shall be applied and lower than the metal seat.

- Open Type

It is the structure to discharge directly into the air with the safety valve operating and the fluid discharged. The operation is relatively smooth as it is not so much effected by the pressure. Provided that it is inappropriate for toxic gas, heating gas, liquid, etc.

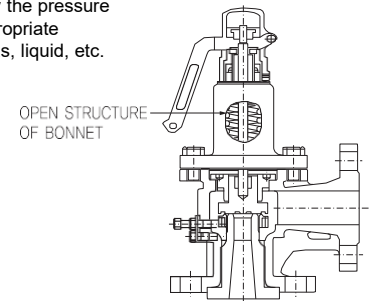


Figure 17
Open Type

- Wing Guide

The guide part of disc is inserted in the seat for compact and the possibility for pressure loss on the discharged part will be less than the top guide type but it tends to have worse discharge coefficient than top guide as the guide in the seat causes resistance to the fluid.

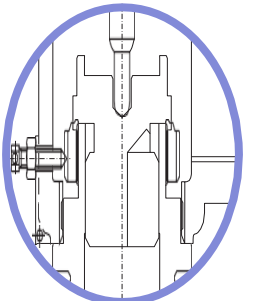


Figure 19
Wing Guide

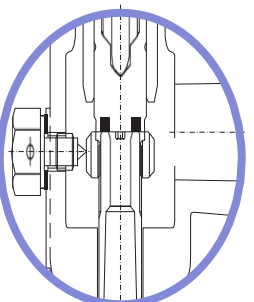


Figure 20
Soft Seat

2-3. Safety Valve and Relief Valve

As explained in the section above, the object of the safety valve is to secure safety of the facility so it is necessary to set up not to operation under normal operation condition. The relief valve is the one with the function to automatically open according to the increase of the liquid pressure at a certain value and it has the object to limit the fixed pressure uniformly. In terms of structure, it is similar to the safety valve so except for special cases, the safety valve can be used as a relief valve. Also, there are some cases to call the structure, which can be used for gas and liquid, as a safety relief valve.

MAINTENANCE MANUAL

3. Meaning of Terms

The meanings of the terms used for the safety valve are defined as follows in KS B6216-1998. Also, besides the standard terms, the frequently used words can be indicated as follows.

• Extract from KS B6216 - 1998

(1) Safety Valve

The valve with the ability to discharge the fluid(vapor and gas) of nominal discharge amount to automatically operate when the entrance pressure reaches the prescribed pressure and to restore to the normal condition again when the pressure decreases.

(2) Fixed Pressure

Pressure indicated on the nameplate as a discharge pressure fixed on the design for the safety valve demanding the discharge pressure and as a discharge starting pressure for the safety valve demand the discharge starting pressure.

(3) Discharge Starting Pressure

As a pressure the safety valve starts to discharge, it is the pressure at the entrance when small amount of fluid(vapor or gas) discharge detected at the exit.

(4) Discharge Pressure

As a pressure at the entrance when the fluid is discharged by the operation of the safety valve, it is the pressure with the lift to the extent, which can be measured or on the condition, which can recognize the consecutive discharge condition.

(5) Pressure to Determine the Nominal Discharge Amount

Pressure to determine the nominal discharge amount, which is prescribed by the attached department.

(6) Discharge Stop Pressure

The pressure at the entrance when the movement becomes 0 as the discharge pressure decreases, the safety valve is closed and the flow of the fluid stops actually.

(7) Pressure of Discharge Difference

Difference between the discharge pressure and discharge stop pressure at the safety valve demanding the discharge pressure and the difference between the discharge start pressure and the discharge stop pressure at the safety valve demanding the discharge start pressure.

(8) Movement

Movement on the axis direction of the disc or valve from the valve closing location to the valve opening location in the course of safety valve discharge.

(9) Nominal Discharge Amount

The discharge amount guaranteeing for each safety valve. Amount to be calculated according to the provision of 5. of KS B 6352(method to measure the discharge coefficient of the safety valve) and the provision of addendum.

(10) Nominal Discharge Coefficient

Coefficient applied to the nominal discharge amount. it shall be determined in the method according to the provision of 4.2.3 of KS B 6352 or the equivalent.

(11) DISC Seat Caliber

Inner diameter of contact of seat and disc.

(12) Discharge Area

As an area of the part to determine the fluid amount to pass through the safety valve, it is the area used for calculation of nominal discharge amount.

(13) Caliber of The Neck

The inner diameter of the narrowest part of the nozzle from the orifice to collect the fluid and the seat side.

(14) Discharge Pressure

Pressure at the exit of the safety valve. There are two kinds of discharge pressure here.

- (a) Pressure generated at the exit of the safety valve by the resistance of exhaust at the discharge of the safety valve.
- (b) Pressure already existing at the exhaust before the safety valve discharges.

Other terms(JIS).

Other Terms(JIS)

Chattering - It means the condition to generate incomplete operation that the disc chatters the seat with the weak up and down vibration on the condition of incomplete discharge or starting the discharge.

Hunting - It means the condition that the disc operation hits the seat seriously by the severe up and down movement due to the local pressure change in the connecting device or pipe resulted from the discharge of the safety valve.

Flutter - It means the condition that the disc vibrates upward and downward to the extent not to chatter the seat in the course of lift when the safety valve is operating.

Flashing - It means the condition that part of the liquid of high temperature evaporates when it is discharged into the air through the relief valve.

Accumulation - It means % of the established pressure of the difference between the pressure to determine the discharge amount and the established pressure.

4. Principle of operatic

Hop orifice is required for the clear operation as a safety valve as indicated in figure 21.

The spring-type safety valve keeps airtight by always pressing the disc to the seat with the power of spring. This spring power is designed and assembled to break the airtight balance at the fixed pressure. After all, at the time when a small amount starts to flow out as the balance becomes upset by the inner pressure to reach the fixed pressure(this phenomenon is referred to as discharge start). At that time, the discharged fluid is saved at the HOP orifice for a moment. At this time, the pressure is cast and the disc pushes upward.(this phenomenon is called to be discharge). The pushing disc become sealed again with the increased spring power as the inner pressure decreases. (this phenomenon is called to be discharge stop)

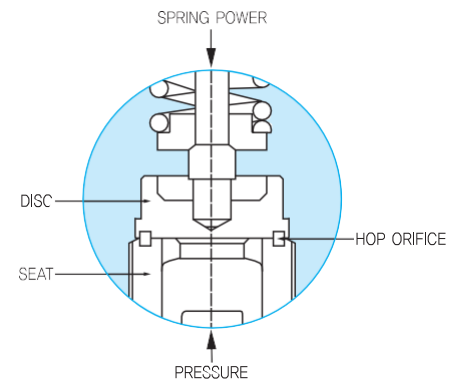


Figure 21
Structure of Seat Part(A)

Figure 22 explains the operation flow.

In this figure, if the time between the discharge start and discharge is short, 'POP operates promptly' That is, as the safety increases, it becomes the safety valve of high efficiency. Besides, there is something with the shape as indicated in Figure 23 to play the role in freely adjusting the HOP orifice and improving other capability but the principle of operation is same as HOP orifice type in Figure 21 as discussed.

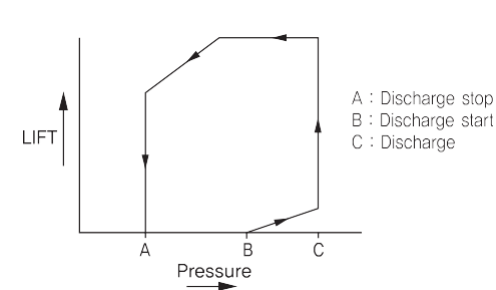


Figure 22
Operation Flow

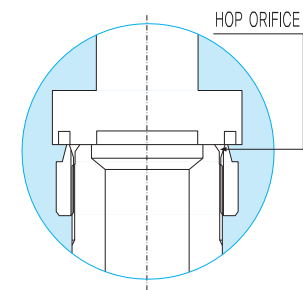
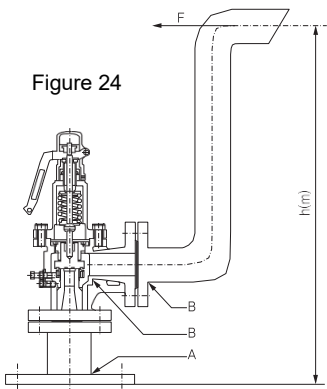


Figure 23
Structure of Seat Part(B)

5. Note for Handling

5-1. Note for Planning

- (a) The safety valve shall be installed vertically on the ground of the place, where it is easy to repair, inspect or adjust it and it is not dangerous to human body at the time of discharge.
- (b) The length in installing the safety valve shall be designed for the decrease of the fluid pressure not to exceed 3% of the discharge pressure at the time of discharge of the safety valve and shall be over the inner diameter of the safety valve entrance.
- (c) The safety valve is rebound in the opposite direction of the exhaust. In the design for installation, it shall have the enough strength against the compression, tensile and bending by rebound.(Figure 24)



Opening and Closing of Rebound According to KSB 6216

F = \frac{W \sqrt{\frac{K T 1}{(K + 1) M}}}{274}

- (d) The length of the safety valve discharge pipe shall be as short as possible making sure that the back pressure at the exit when the safety valve discharges, do not exceed 10% of the discharge pressure. Also, it shall be more than the exit diameter of the safety valve.
- (e) If uniting 2 safety valve discharge pipes or more, the angle shall be less than 45 (Y-type) and the pipe are of the jointed bottom shall be more than the sum of each pipe area.
- (f) If inevitably installing the safety valve in the pipe, the length of the joint pipe shall be as short as possible.
- (g) The open drainage shall be installed at the bottom lower than the valve seat in the valve body exit of the safety valve attaching the discharge pipe. Provided that the drainage of discharge pipe can be shared if the designation diameter is less than 50mm. The drainage of the safety valve, whose valve body is cast iron or cast steel and whose designation diameter exceeds 65mm, shall comply with the provision of KSB 0222. (official taper screw)

5-2. Note for Piping Installation

- (a) If installing the screw-type safety valve, do not turn it by holding the body but make sure to connect it by using the hexagonal side of the screw. If turning and connecting it by holding the body, it may cause leakage, etc. Also, bind the seal tape out of the edge so it is not entangled on the edge of the screw. If it is entangled on the edge, the tape will be cut when draining to cause leakage.
- (b) Precise lapping processing has been done to increase the sealing performance on the seat of the safety valve. If foreign substances such as dust or scale contact the seat in the operation of the safety valve, it may cause the leakage of the seat. Therefore, clean the tank and piping thoroughly before installing the safety valve and carry out the air purge.

6. Repair and Inspection

6-1. Please perform the repair and inspection at the same interval as the appearance inspection of the facility.

(a) Appearance inspection

- 1 - Check whether the valve box is rotten or cracked.
- 2 - Whether the safety valve leaks under the normal pressure. (check the leakage noise and temperature change)
- 3 - Whether the installation section of the safety valve leaks
- 4 - Check any abnormal vibration of the facility
- 5- Check the opening of the main valve of the safety valve.(Keep it opened)

6-2. Safety Inspection

(a) The regular inspection shall be done every 6 months and sometimes check the following items.

- 1 - Condition on the flange side.(corrosion, damage, etc)
- 2 - Attachment condition of the foreign substance, scale, etc at the passage of the entrance.
- 3 - Corrosion and damage of the valve box
- 4 - Piping inspection

(b) In principle, perform the inspection of item a of 6-1 above after the popping of the safety valve and immediately repair any defect.

(c) Operation pressure check

1 - Check whether there is no change by comparing the discharge start pressure or discharge pressure and stop pressure with the values indicated on the nameplate of the safety valve for the operation pressure.

2 - At this time, please use air or incombustible gas in principle for the fluid.

(d) Seat sealing check

1 - Check whether there is no leakage at 90% of the fixed pressure.

7. Adjustment

If it is not necessary to adjust the discharge pressure or discharge difference pressure, it can be done as follows.

7-1. Adjustment of discharge pressure

(Note) 1 - At this time, make sure that the pressure is removed.

2 - It is dangerous if the safety valve is operated while adjusting it, so do not make the face upward nor stand on the side of the exhaust.

The discharge pressure is strictly adjusted in the performance inspection by our company but there are some cases of some differences according to the variation of conditions such as safety valve design location, distance to the pressure gauge, etc. The tolerance of the discharge pressure for 'hot air and gas spring safety valve' of KS B 6216-1998 is prescribed as indicated in the following table.

Table 1

For Hot Air	
Fixed Pressure kgf / cm² {MPa}	Tolerance kgf / cm² {MPa}
Less than {0.5}	± 0.14 {0.014}
From 5 {0.5} to 23 {2.3}	± (3% of the fixed pressure)
From 23 {2.3} to 70 {7.0}	± 0.7 {0.07}
More than 70 {7.0}	± (1% of the fixed pressure)

for Gas

From the fixed pressure to 1.1 times of the fixed pressure

- (a) Closed-type safety valve (Figure 25)
- 1 - Detach the sealing and CAP A(Figure 25)
- 2 - Loosen the stop nut C(Figure 25)
- 3 - The discharge pressure can be adjusted by turning the adjusting screw B(Figure 25) (turn it clockwise on the top to increase the discharge pressure and counterclockwise to decrease it)
- 4 - Fasten the stop nut C(Figure 25) after the adjustment. (At this time, make sure that the adjusting screw B(Figure 25) do not turn together)
- 5 - Also the adjustment scope shall be less than ±10%.

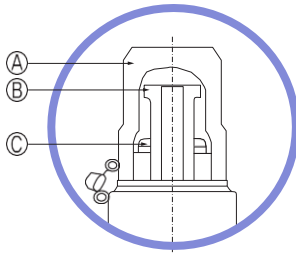


Figure 25

for Gas

From the fixed pressure to 1.1 times of the fixed pressure

- (a) Closed-type safety valve (Figure 25)
- 1 - Detach the sealing and CAP A(Figure 25)
 - 2 - Loosen the stop nut C(Figure 25)
 - 3 - The discharge pressure can be adjusted by turning the adjusting screw B(Figure 25)
(turn it clockwise on the top to increase the discharge pressure and counterclockwise to decrease it)
 - 4 - Fasten the stop nut C(Figure 25) after the adjustment.
(At this time, make sure that the adjusting screw B(Figure 25) do not turn together)
 - 5 - Also the adjustment scope shall be less than $\pm 10\%$.

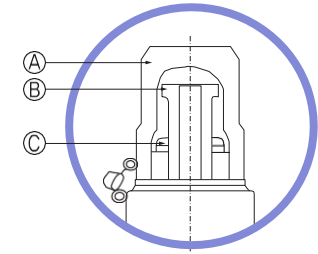


Figure 25

(b)Lever-type safety valve (Figure 26)

- 1 - Cut the sealing, pull the rivet C(Figure 26) out and remove handle B(Figure 26).
- 2 - Remove the hexagonal stop screw D(Figure 26) and then the lever cap A (Figure 26)
- 3 - After this, the discharge pressure can be adjusted just like the closed type safety valve of 7-1.a.
- 4 - In reassembling, make sure the lever B(Figure 26) is 1mm apart from the screw washer.

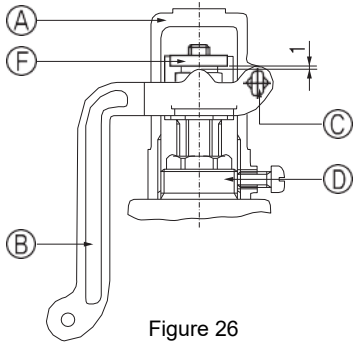


Figure 26

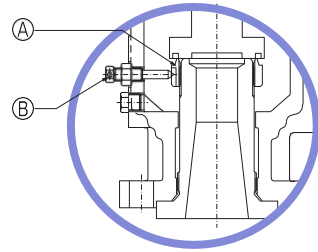


Figure 27

7-2. Adjustment of Discharge Difference Pressure (Figure 27)

- (Note)
- 1 - At this time, make sure that the pressure is removed.
 - 2 - It is dangerous if the safety valve is operated while adjusting it, so do not make the face upward nor stand on the side of the exhaust. As for those with the device, which can adjust the discharge difference pressure by type.
(adjusting ring section), it can be adjusted as follows.

8. Cause and Measure of Trouble

The cause of the safety valve trouble are various according to the conditions such as the surrounding atmospheres of the using fluid and it is difficult to explain all of them so we select and describe the representative items among them here.

8-1. Seat Leakage

The sealing performance of the safety valve depends of technology mostly. There were some leaky safety valves before the present lapping technology was developed but as the lapping technology greatly develops lately, it becomes possible to manufacture the safety valve of no leakage. However, if not paying sufficient attention to handling it, the leakage may be caused from the beginning of the use.(refer to 5-2 for the note for piping installation)

- (a) By foreign substances
- If not thoroughly performing the air purge for the piping, or in case of trial run with the sludge inserted in the fluid, there are some cases to put foreign substances in the seat and disc. At this time, they may be removed by manual operation simply but even after the operation, the seal side shall be sufficiently checked against any damage or foreign substances. In case of this damage or foreign substances, it is hard to handle in the user's position so it is necessary to return it to manufacturer promptly.

- (b) Pipe pressure
- 1 - Personal cause
- When doing the secondary piping for the safety valve, if turning the safety valve severely, the seat may spin in the aspect of the safety valve structure. At this time, the sealing side gets damaged or the location of the adjustment ring changes, which may cause the weakness of the functioning. In this case, there are cases that the user repair or it is necessary to return to the manufacturer for replacement. It is required to sufficiently recognize and judge them and promptly handle it.

- 2 - Inner stress cause
- There is a case to have an adverse influence on the safety valve by the heat stress or remaining stress of the pipe. The problems of this is almost similar as the a forementioned personal causes but it is important to lay pipes, which can completely absorb the stress after the piping work.

- (c) Balance of use stress and fixed stress
- We explain the procedure from voltage boosting to discharge stop in the operation principle of the safety valve. As explained here, there is a phenomenon of discharge start before the discharge of the safety valve. If the use pressure considerably approaches to the fixed pressure, the phenomenon of discharge start is repeated as stated above, which may cause a disaster. It is desirable to design the use pressure to be less than 85% of the fixed pressure.

8-2. Incomplete operation

The phenomenons of hunting, chattering and flutter in the explanation of terms occur if the discharge stop is not too large(considerably close to the fixed pressure) or if applying not too much back pressure when discharging the fluid(if it becomes difficult to exhaust the discharged fluid). Naturally, the sealing side or frictional section gets damaged by the upward or downward movement of the disc. It is good to design the back pressure on the discharge side not to exceed 10% of the discharge pressure.

8-3. Corrosion

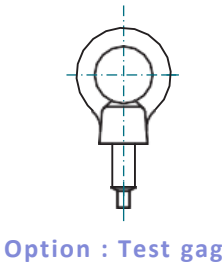
The problem of corrosion can be solved by sufficient prior discussion between users and manufacturer in general but there are some cases that a fluid of non corrosion or materials, which are not corroded normally, corrode under some situations.(for example, electrolytic corrosion). It is important to sufficiently check it when inspecting for repair of the safety valve and to immediately handle it.

Numbering System

KST Series – Screw Type, Low Lift Type

Selecting and specifying KSM Safety & Relief Valves is simple using the numbering system that follows. Each digit of the part number has a distinct significance. The digits describe the basic valve series, valve type, inlet / outlet type and connection type.

Model	Type	Lever	Use Condition
KST - 11	Conventional	No	Steam, Air & Gas, Vapor, Water & Liquid
KST - 12	Conventional	Yes	Steam, Air & Gas, Vapor



Option : Test gag

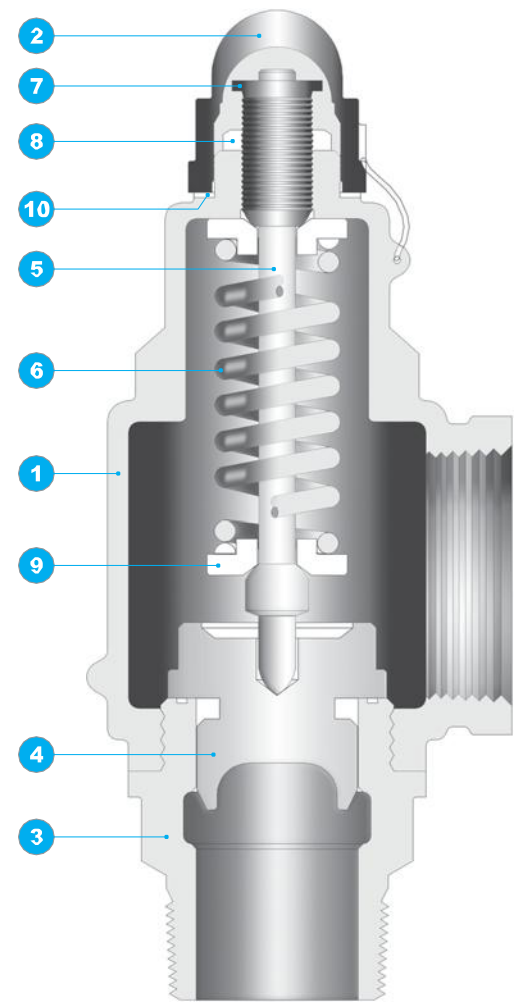
Example to Describe Model (KST-11, 15A M.PT x 20A F.PT)

K	S	T	11	15A	M	PT	20A	F	PT
Company Code	Valve Name	Connection Type	Valve Type	Inlet Size	Inlet Screw Connection	Inlet Connection	Outlet Size	Outlet Screw Connection	Outlet Connection
K : Korea Special Metal	S : Safety & Relief Valve	T : Screw Type	11: No lever 12 : Lever	15A, 20A, 25A 32A, 40A, 50A	M : Male F : Female	PT : Pipe Taper Thread NPT : National Pipe Taper Thread	20A, 25A, 32A 40A, 50A	M : Male F : Female	PT : Pipe Taper Thread NPT : National Pipe Taper Thread

BILL OF MATERIALS

Series KST – 11 (Size 15A/20A/32A)

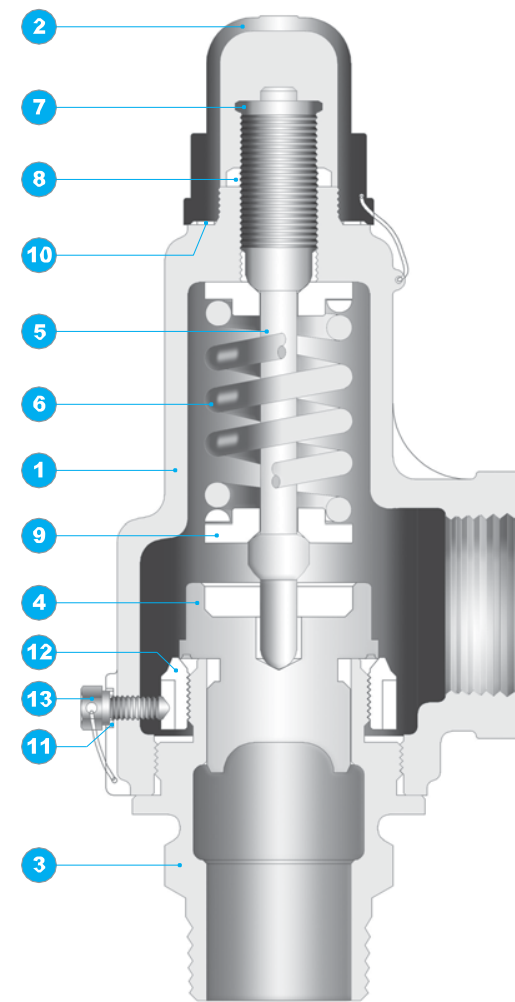
This valves are small, compact and high-performance valves widely used for small boiler, thermal expansion and the petrochemical industry as well as chemical and general industries and cover the wide pressure range, from low to high, of steam, air and gas, vapor, water and liquid.



No.	Part Name	Standard Material
1	Body	Bronze Cast Steel Stainless Steel
2	Cap	Bronze Cast Steel Stainless Steel
3	Seat	Bronze Stainless Steel
4	Disc	Brass Stainless Steel
5	Spindle	Brass Carbon Steel Stainless Steel
6	Spring	Carbon Steel Stainless Steel
7	Adjusting Screw	Brass Carbon Steel Stainless Steel
8	Adjusting Screw Nut	Brass Carbon Steel Stainless Steel
9	Spring Guide	Brass Carbon Steel Stainless Steel
10	Gasket	Non-Asbestos PTFE Graphite

Series KST – 11 (Size 40A/50A)

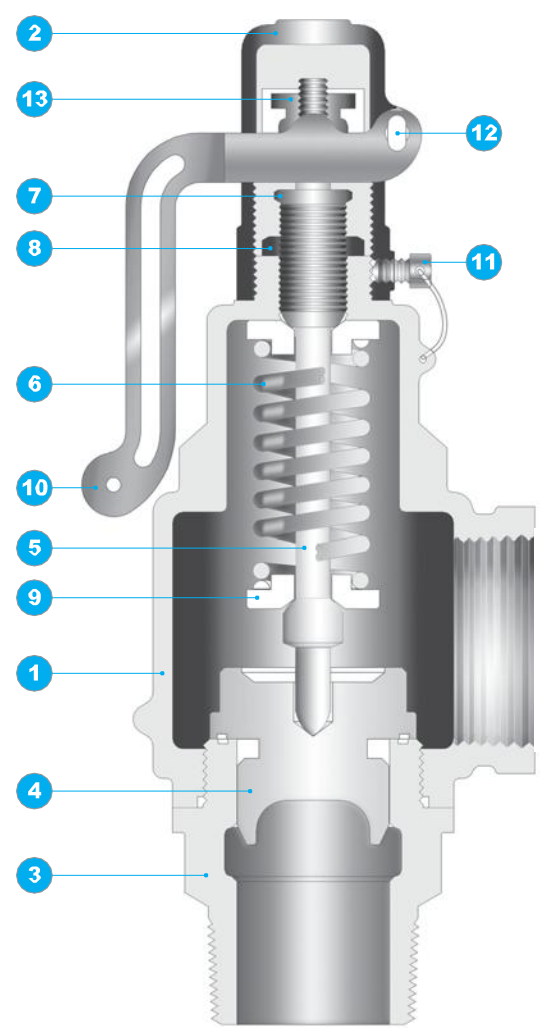
This valves are small, compact and high-performance valves widely used for small boiler, thermal expansion and the petrochemical industry as well as chemical and general industries and cover the wide pressure range, from low to high, of steam, air and gas, vapor, water and liquid.



No.	Part Name	Standard Material
1	Body	Ductile Cast Steel Stainless Steel
2	Cap	Bronze Cast Steel Stainless Steel
3	Seat	Bronze Stainless Steel
4	Disc	Brass Stainless Steel
5	Spindle	Carbon Steel Stainless Steel
6	Spring	Carbon Steel Stainless Steel
7	Adjusting Screw	Brass Carbon Steel Stainless Steel
8	Adjusting Screw Nut	Brass Carbon Steel Stainless Steel
9	Spring Guide	Brass Carbon Steel Stainless Steel
10	Gasket	Non-Asbestos PTFE Graphite
11	Gasket	Non-Asbestos PTFE Graphite
12	Blow Down Ring	Brass Stainless Steel
13	Lock Bolt	Brass Carbon Steel Stainless Steel

Series KST – 12 (Size 15A/20A/32A)

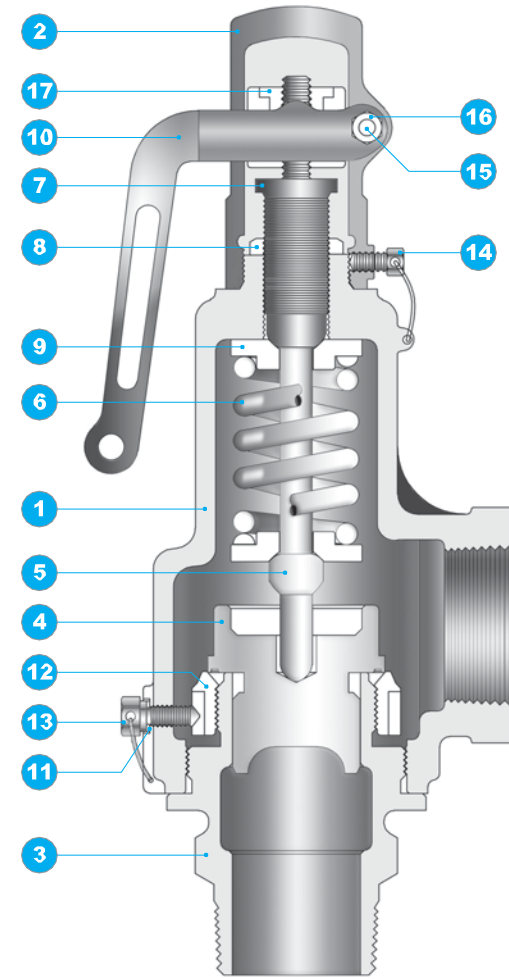
This valves are small, compact and high-performance valves widely used for small boiler, thermal expansion and the petrochemical industry as well as chemical and general industries and cover the wide pressure range, from low to high, of steam, air and gas, vapor.



No.	Part Name	Standard Material
1	Body	Bronze Cast Steel Stainless Steel
2	Cap	Bronze Cast Steel Stainless Steel
3	Seat	Bronze Stainless Steel
4	Disc	Brass Stainless Steel
5	Spindle	Brass Carbon Steel Stainless Steel
6	Spring	Carbon Steel Stainless Steel
7	Adjusting Screw	Brass Carbon Steel Stainless Steel
8	Adjusting Screw Nut	Brass Carbon Steel Stainless Steel
9	Spring Guide	Brass Carbon Steel Stainless Steel
10	Lever	Bronze
11	HEX. Bolt	Brass
12	Pin	Brass
13	Lifting Washer	Brass

Series KST – 12 (Size 40A/50A)

This valves are small, compact and high performance valves widely used for small boiler, thermal expansion and the petrochemical industry as well as chemical and general industries and cover the wide pressure range, from low to high, of steam, air and gas, vapor.

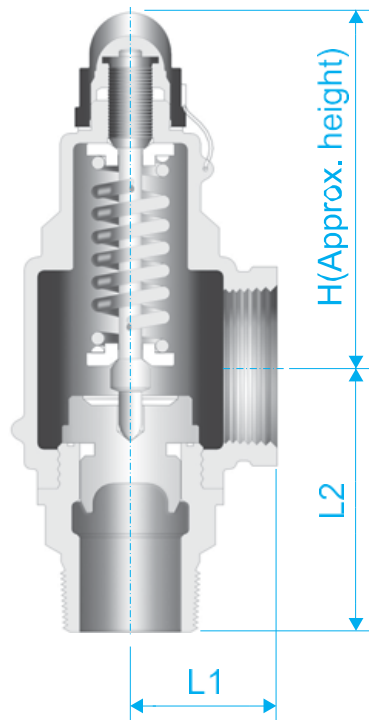


No.	Part Name	Standard Material
1	Body	Ductile Cast Steel Stainless Steel
2	Cap	Bronze Cast Steel Stainless Steel
3	Seat	Bronze Stainless Steel
4	Disc	Brass Stainless Steel
5	Spindle	Carbon Steel Stainless Steel
6	Spring	Carbon Steel Stainless Steel
7	Adjusting Screw	Brass Carbon Steel Stainless Steel
8	Adjusting Screw Nut	Brass Carbon Steel Stainless Steel
9	Spring Guide	Brass Carbon Steel Stainless Steel
10	Lever	Bronze
11	Gasket	Non-Asbestos PTFE Graphite
12	Blow Down Ring	Brass Stainless Steel
13	Lock Bolt	Brass Carbon Steel Stainless Steel
14	HEX. Bolt	Brass
15	HEX. Bolt	Carbon Steel
16	Nut	Carbon Steel
17	Lifting Washer	Brass

GENERAL DIMENSION

STANDARD VALVE SELECTION TABLE

KST - 11



Inlet / Outlet Screw - PT

Low Lift Type

(Unit : mm)

Size Inlet x Outlet	Discharge Area (mm ²)	Center to Face		Approx. Height H	Approx. Height (kg)
		L1	L2		
15A x 20A	25.434	34	53	82	0.85
20A x 20A	25.434	34	53	82	0.90
25A x 25A	45.216	40	65	96	1.25
32A x 32A	75.4385	45	78	105	1.70
40A x 40A	113.354	64	104	155	3.90
50A x 50A	180.864	83	113	192	7.20

Center to Face Tolerance : $\pm 1.5\text{mm}$

Inlet / Outlet Screw - PT or NPT

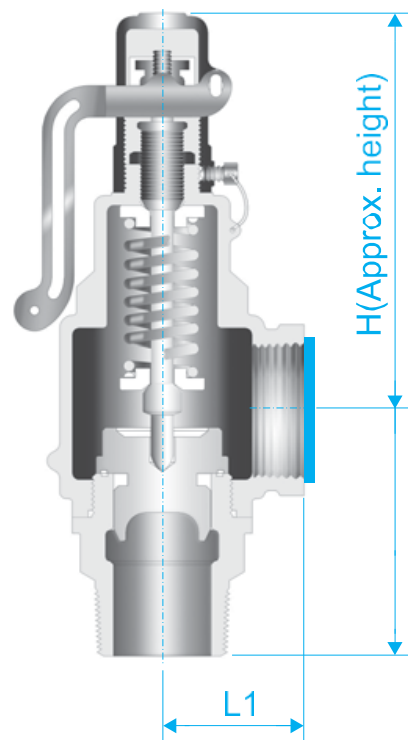
Full Lift Type

(Unit : mm)

Size Inlet x Outlet	Orifice Area (mm ²)	Center to Face		Approx. Height H	Approx. Height (kg)
		L1	L2		
15A x 20A	78.5	43	66	116	1.48
20A x 25A	78.5	45	69	116	1.52
25A x 40A	132.665	64	104	155	4.10

Center to Face Tolerance : $\pm 1.5\text{mm}$

KST - 12



Inlet / Outlet Screw - PT

Low Lift Type

(Unit : mm)

Size Inlet x Outlet	Discharge Area (mm ²)	Center to Face		Approx. Height H	Approx. Height (kg)
		L1	L2		
15A x 20A	25.434	34	53	101	0.90
20A x 20A	25.434	34	53	101	0.95
25A x 25A	45.216	40	65	117	1.30
32A x 32A	75.4385	45	78	126	1.90
40A x 40A	113.354	64	104	184	4.15
50A x 50A	180.864	83	113	221	7.45

Center to Face Tolerance : $\pm 1.5\text{mm}$

Inlet / Outlet Screw - PT or NPT

Full Lift Type

(Unit : mm)

Size Inlet x Outlet	Orifice Area (mm ²)	Center to Face		Approx. Height H	Approx. Height (kg)
		L1	L2		
15A x 20A	78.5	43	66	126	1.64
20A x 25A	78.5	45	69	126	1.69
25A x 40A	132.665	64	104	184	4.10

Center to Face Tolerance : $\pm 1.5\text{mm}$

Low Lift Type

(Unit : mm)

Materials			Type	Valve Size	End Connection	
Body	Seat & Disc	Spring		Inlet x Orifice DIA. x Outlet	Inlet (Male)	Outlet (Female)
Bronze	Bronze & Brass	Carbon Steel	KST - 11	15A x 12.5 x 20A	PT	PT
Carbon Steel	Stainless Steel	Stainless Steel	KST - 12	20A x 18 x 25A		
Stainless Steel {32A and Smaller}				25A x 24 x 25A 32A x 31 x 32A		
Ductile	Bronze & Brass	Carbon Steel	KST - 11	40A x 38 x 40A		
Carbon Steel Stainless Steel {40A and Larger}	Stainless Steel	Stainless Steel	KST - 12	50A x 48 x 50A		

Full Lift Type

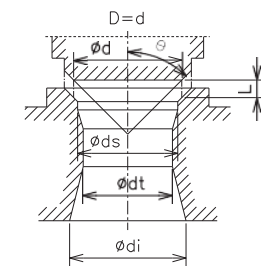
(Unit : mm)

Materials			Type	Valve Size	End Connection	
Body	Seat & Disc	Spring		Inlet x Orifice DIA. x Outlet	Inlet (Male)	Outlet (Female)
Carbon Steel	Stainless Steel	Carbon Steel	KST - 11	15A x 10 x 20A	PT	PT
Stainless Steel		Stainless Steel	KST - 12	20A x 10 x 25A	or	or
{32A and Smaller}				25A x 13 x 40A	NPT	NPT

If at particular over pressure the valve attains high lift, the discharge area is established by measurement of the inlet passage at its least diameter. Some safety valves operate at low lift, in which case the controlling flow area is the so-called curtain area. This area is the product of the circumference of the minimum passage through the valve seat and valve lift.

Formulas for the Curtain Area are :

$$\begin{aligned}
 &= \pi DL \quad (\text{Flat Seat}) \quad - \text{Lift Type} \\
 &= \pi DL \sin \theta \quad (\text{Conical Seat}) \quad - \text{Lift Type} \\
 &= \pi / 4 \cdot d_t^2 \quad (\text{Full Lift Type})
 \end{aligned}$$



In case
 $d_s < d$
as Conical Seat

Where.

A : Effective Discharge Area (mm²)

D : Diameter of Disc Seat Opening (mm)

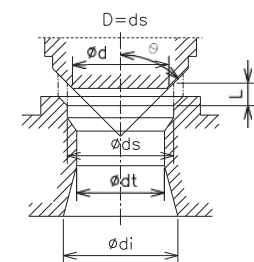
θ : Angle between the discharging part and the valve axis $\rightarrow \sin 45^\circ = 0.707$

d : Inside Diameter of Disc Seat Face on the Disc Side (mm)

d t : Throat Diameter (mm)

d s : Bore of Disc Seat Face (mm)

d i : Diameter of in Take Opening for Steam or Gas (Nominal Diameter)



In case
 $d_s > d$
as Conical Seat

Fig. 1
Effective Discharge Area and Diameter of Disc Seat Opening

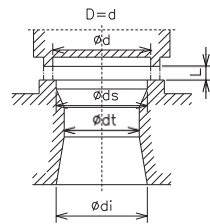
CALCULATION OF SAFETY & RELIEF VALVE CAPACITY

Code : KS B 6216 or KOSHA for Steam

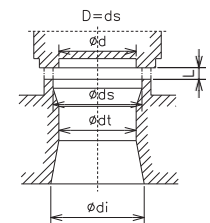
$$W = 5.246 \times C \times K_d \times A \times (P+0.1) \times 0.9$$

Nomenclature

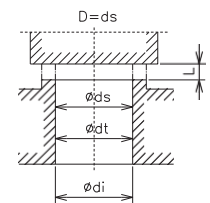
W	Capacity of Steam, kg / h
A	Effective Discharge Area, mm ² Low Lift Type : A = π DL D = Diameter of Disc Seat (mm) L = Lift (mm)
P	For Set Pressure ≤ 0.1 MPa P = Set Pressure + 0.02 MPa For Set Pressure > 0.1 MPa P = Set Pressure x 1.03
K d	Coefficient of Discharge (Fig.3, page 12)
C	Superheat Correction Factor



In case
d s < d
as Flat Seat



In case
d s > d
as Flat Seat



In case
d s = d t
as Flat Seat

Fig. 2
Effective Discharge Area and Diameter of Disc Seat Opening

Capacity Table of Sat. Steam

Low Lift Type		(Unit : kg / h)					
Size		15A	20A	25A	32A	40A	50A
Dia. of Disc Seat (mm)		18	18	24	31	38	48
Lift (mm)		0.45	0.45	0.6	0.775	0.95	1.2
0.2	0.02	16.5	16.5	29.3	48.9	73.5	117
0.3	0.029	17.5	17.5	31.2	52.1	78.3	124
0.5	0.049	19.9	19.9	35.4	59.1	88.8	141
0.7	0.069	22.2	22.2	39.6	66.1	99.3	158
0.9	0.088	24.5	24.5	43.6	72.7	109	174
1	0.098	25.7	25.7	45.7	76.2	114	182
1.2	0.118	26.1	26.1	46.4	77.4	116	185
1.5	0.147	29.6	29.6	52.7	87.9	132	210
1.7	0.167	32	32	57	95.1	142	228
2	0.196	35.5	35.5	63.2	105	158	253
2.2	0.216	38	38	67.6	112	169	270
2.5	0.245	41.5	41.5	73.8	123	185	295
2.7	0.265	43.9	43.9	78.1	130	196	312
3	0.294	47.5	47.5	84.4	140	211	337
3.2	0.314	49.9	49.9	88.7	148	222	355
3.5	0.343	53.4	53.4	95	158	238	380
3.7	0.363	55.8	55.8	99.3	165	249	397
4	0.392	59.4	59.4	105	176	264	422
4.2	0.412	61.8	61.8	109	183	275	439
4.5	0.441	65.3	65.3	116	193	291	464
4.7	0.461	67.7	67.7	120	201	302	482
5	0.49	71.3	71.3	126	211	317	507
5.2	0.51	73.7	73.7	131	218	328	524
5.5	0.539	77.2	77.2	137	229	344	549
5.7	0.559	79.6	79.6	141	236	355	566
6	0.588	83.2	83.2	147	246	370	591
6.2	0.608	85.6	85.6	152	254	381	608
6.5	0.637	89.1	89.1	158	264	397	634
6.7	0.657	91.5	91.5	162	271	408	651
6.9	0.676	93.8	93.8	166	278	418	667
7	0.686	95.1	95.1	169	282	423	676
7.2	0.706	97.5	97.5	173	289	434	693
7.5	0.735	101	101	179	299	450	718
7.7	0.755	103	103	183	306	461	735
8	0.784	107	107	190	317	476	761
8.2	0.804	109	109	194	324	487	778
8.5	0.833	112	112	200	335	503	803
8.7	0.853	115	115	205	342	514	820
9	0.882	118	118	211	352	530	845
9.2	0.902	121	121	215	359	540	862
9.5	0.931	124	124	221	370	556	887
9.7	0.951	127	127	226	377	567	905
10	0.98	130	130	232	388	583	930
10.2	1	133	133	236	395	593	947
10.5	1.029	136	136	243	405	609	972
Kg / cm ² g MPag		-					
Pressure		-					

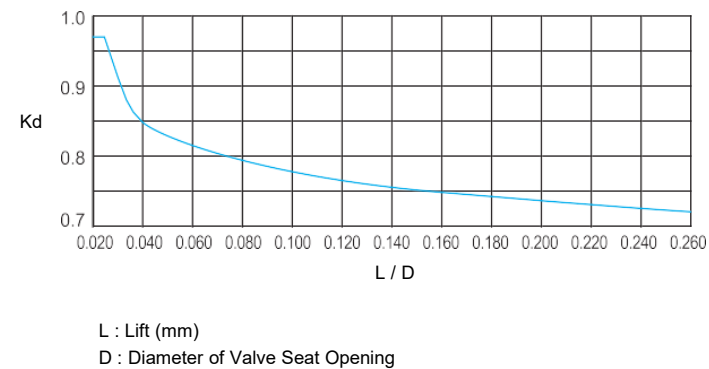
Code : KS B 6216 or KOSHA for Air

$$W = \frac{C \times K_d \times A \times P_1 \times (M)^{1/2}}{(Z \times T)^{1/2}} \times 0.9$$

Nomenclature

W	Capacity of Air, kg / h
A	Effective Discharge Area, mm ² Low Lift Type : A = π DL D = Diameter of Disc Seat (mm) L = Lift (mm)
P1	Upstream Relieving Pressure Absolute, MPa
M	Molecular Weight Air = 28.96
T	Absolute Temperature of Gas, K
C	Coefficient Determined by Ratio of the Specific Heats of Gas
K d	Coefficient of Discharge (Fig. 3, page 12)
Z	Compressibility Factor

Figure 3 Coefficient of Discharge Kd



Capacity Table of Air

Low Lift Type		(Unit : kg / h)					
Size		15A	20A	25A	32A	40A	50A
Dia. of Disc Seat (mm)		18	18	24	31	38	48
Lift (mm)		0.45	0.45	0.6	0.775	0.95	1.2
0.2	0.02	23.4	23.4	41.7	69.6	104	166
0.3	0.029	25.3	25.3	45	75.2	113	180
0.5	0.049	29.5	29.5	52.5	87.6	131	210
0.7	0.069	33.7	33.7	60	100	150	240
0.9	0.088	37.7	37.7	67	111	168	268
1	0.098	39.8	39.8	70.8	118	177	283
1.2	0.118	44	44	78.2	130	196	313
1.5	0.147	50.1	50.1	89.1	148	223	356
1.7	0.167	54.3	54.3	96.5	161	242	386
2	0.196	60.4	60.4	107	179	269	429
2.2	0.216	64.6	64.6	114	191	287	459
2.5	0.245	70.6	70.6	125	209	315	502
2.7	0.265	74.8	74.8	133	222	333	532
3	0.294	80.9	80.9	143	240	360	575
3.2	0.314	85.1	85.1	151	252	379	605
3.5	0.343	91.2	91.2	162	270	406	648
3.7	0.363	95.4	95.4	169	283	425	678
4	0.392	101	101	180	301	452	722
4.2	0.412	105	105	187	313	471	751
4.5	0.441	111	111	198	331	498	795
4.7	0.461	116	116	206	344	517	825
5	0.49	122	122	217	362	544	868
5.2	0.51	126	126	224	374	562	898
5.5	0.539	132	132	235	392	590	941
5.7	0.559	136	136	242	405	608	971
6	0.588	142	142	253	423	635	1014
6.2	0.608	146	146	261	435	654	1044
6.5	0.637	152	152	271	453	681	1087
6.7	0.657	157	157	279	466	700	1117
6.9	0.676	161	161	286	477	718	1146
7	0.686	163	163	290	484	727	1160
7.2	0.706	167	167	297	496	746	1190
7.5	0.735	173	173	308	514	773	1234
7.7	0.755	177	177	315	527	792	1263
8	0.784	183	183	326	545	819	1307
8.2	0.804	188	188	334	557	837	1337
8.5	0.833	194	194	345	575	865	1380
8.7	0.853	198	198	352	588	883	1410
9	0.882	204	204	363	606	910	1453
9.2	0.902	208	208	370	618	929	1483
9.5	0.931	214	214	381	636	956	1526
9.7	0.951	218	218	389	649	975	1556
9.9	0.97	222	222	396	661	993	1584
10	0.98	224	224	399	667	1002	1599
10.2	1	229	229	407	679	1021	1629
Kg / cm ² g MPag		-					
Pressure		-					

Code : KS B 6216 or KOSHA for Nitrogen

W = $\frac{C \times K_d \times A \times P_1 \times (M)^{1/2} \times 0.9}{(Z \times T)^{1/2}}$

Nomenclature

W	Capacity of Nitrogen, kg / h
A	Effective Discharge Area, mm ² Low Lift Type : A = TT DL D = Diameter of Disc Seat (mm) L = Lift (mm)
P1	Upstream Relieving Pressure Absolute, MPa
M	Molecular Weight Nitrogen = 28
T	Absolute Temperature of Gas, K
C	Coefficient Determined by Ratio of the Specific Heats of Gas
Kd	Coefficient of Discharge (Fig. 3, page 12)
Z	Compressibility Factor

Capacity Table of Nitrogen

Low Lift Type		(Unit : kg / h)					
Size		15A	20A	25A	32A	40A	50A
Dia. of Disc Seat (mm)		18	18	24	31	38	48
Lift (mm)		0.45	0.45	0.6	0.775	0.95	1.2
0.2	0.02	23	23	41	68.4	102	164
0.3	0.029	24.9	24.9	44.3	73.9	111	177
0.5	0.049	29	29	51.6	86.2	129	206
0.7	0.069	33.1	33.1	59	98.4	147	236
0.9	0.088	37.1	37.1	65.9	110	165	263
1	0.098	39.1	39.1	69.6	116	174	278
1.2	0.118	43.3	43.3	76.9	128	192	307
1.5	0.147	49.2	49.2	87.6	146	219	350
1.7	0.167	53.4	53.4	94.9	158	238	379
2	0.196	59.4	59.4	105	176	264	422
2.2	0.216	63.5	63.5	112	188	283	451
2.5	0.245	69.5	69.5	123	206	309	494
2.7	0.265	73.6	73.6	130	218	328	523
3	0.294	79.6	79.6	141	236	354	566
3.2	0.314	83.7	83.7	148	248	373	595
3.5	0.343	89.7	89.7	159	266	399	638
3.7	0.363	93.8	93.8	166	278	418	667
4	0.392	99.8	99.8	177	296	445	710
4.2	0.412	103	103	184	308	463	739
4.5	0.441	109	109	195	326	490	781
4.7	0.461	114	114	202	338	508	811
5	0.49	120	120	213	356	535	853
5.2	0.51	124	124	220	368	553	883
5.5	0.539	130	130	231	386	580	925
5.7	0.559	134	134	238	398	598	955
6	0.588	140	140	249	416	625	997
6.2	0.608	144	144	256	428	643	1027
6.5	0.637	150	150	267	446	670	1069
6.7	0.657	154	154	274	458	688	1098
6.9	0.676	158	158	281	470	706	1126
7	0.686	160	160	285	476	715	1141
7.2	0.706	164	164	292	488	733	1170
7.5	0.735	170	170	303	506	760	1213
7.7	0.755	174	174	310	518	778	1242
8	0.784	180	180	321	536	805	1285
8.2	0.804	184	184	328	548	823	1314
8.5	0.833	190	190	339	566	850	1357
8.7	0.853	194	194	346	578	869	1386
9	0.882	200	200	357	596	895	1429
9.2	0.902	205	205	364	608	914	1458
9.5	0.931	211	211	375	626	940	1501
9.7	0.951	215	215	382	638	959	1530
9.9	0.97	219	219	389	649	976	1558
10	0.98	221	221	393	656	985	1573
10.2	1	225	225	400	668	1004	1602
Kg / cm ² g MPag		-					
Pressure		-					

Code : HPGCL for Water

W = $\frac{K_d \times K_w \times K_c \times K_p \times A \times (1.25P - P_b)^{1/2}}{11.78 \times (G)^{1/2}} \times 60$

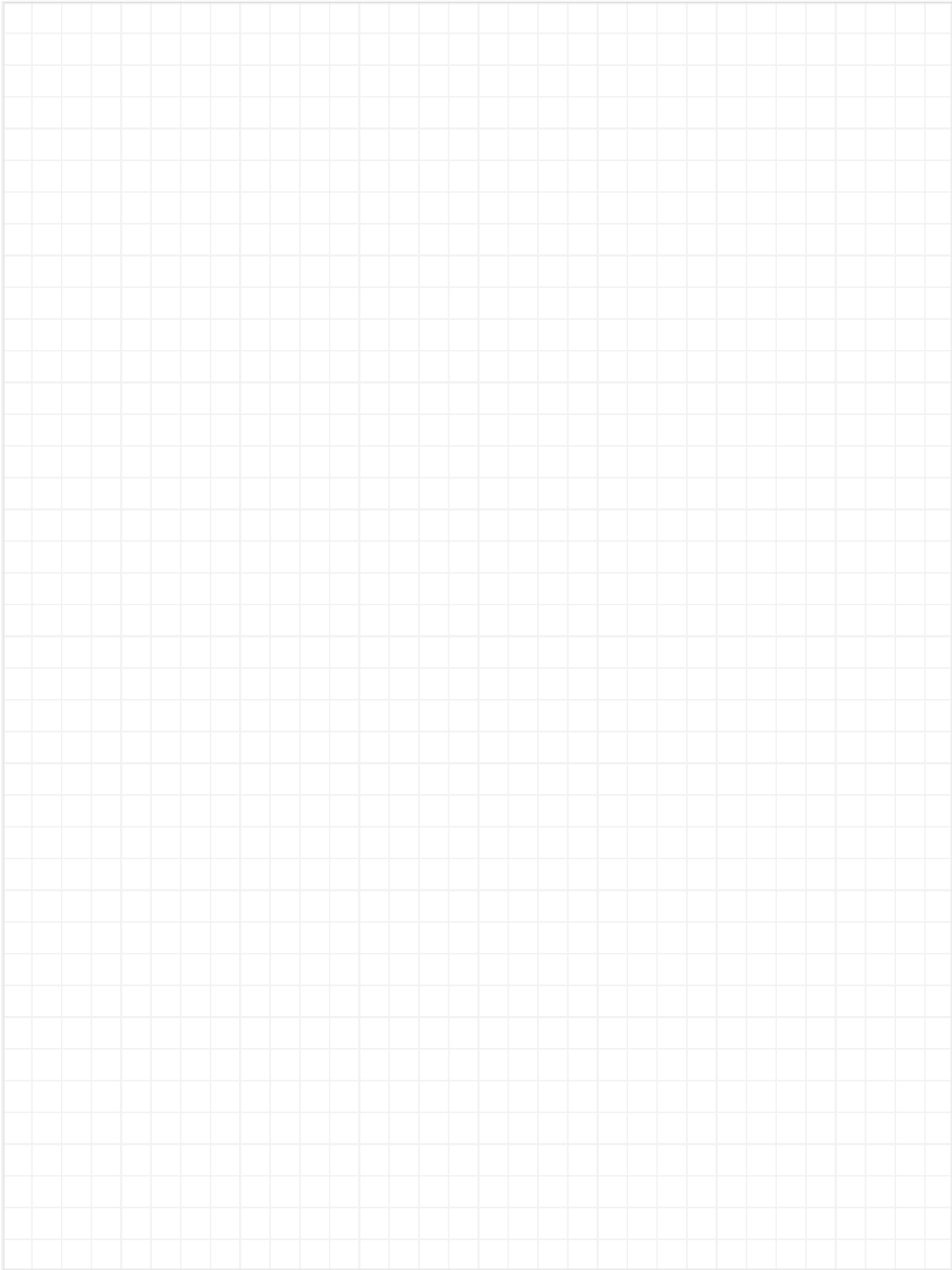
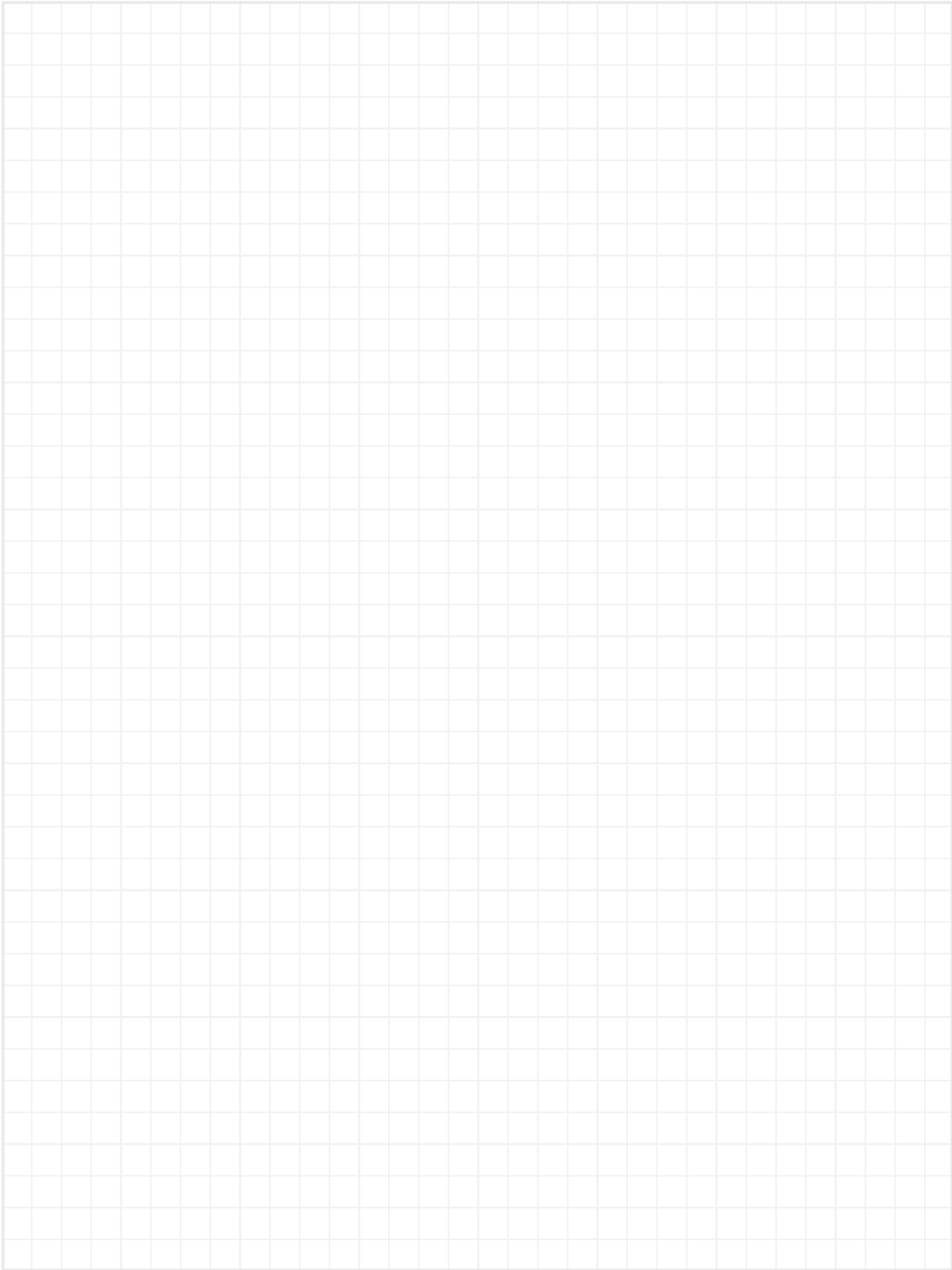
Nomenclature

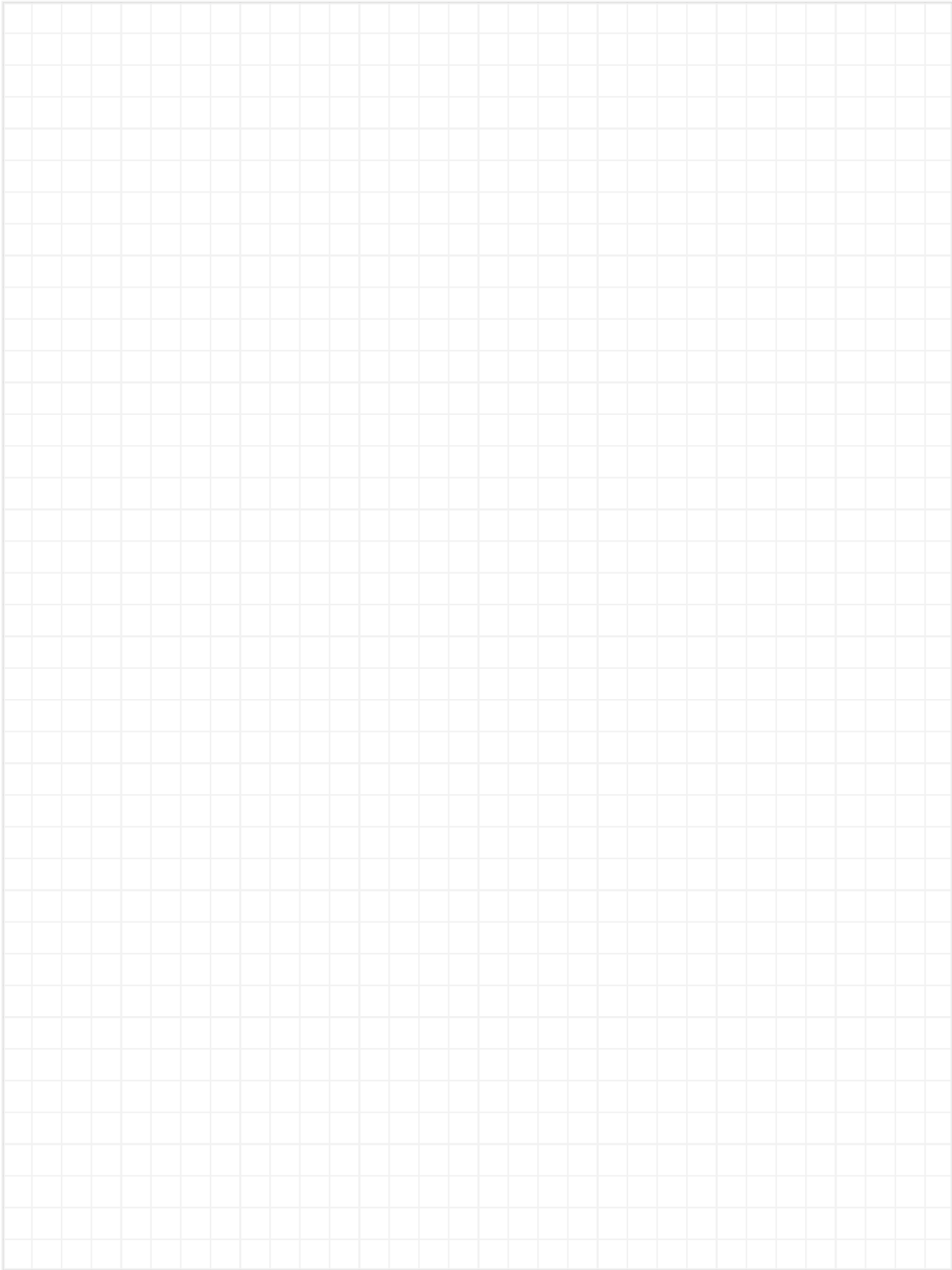
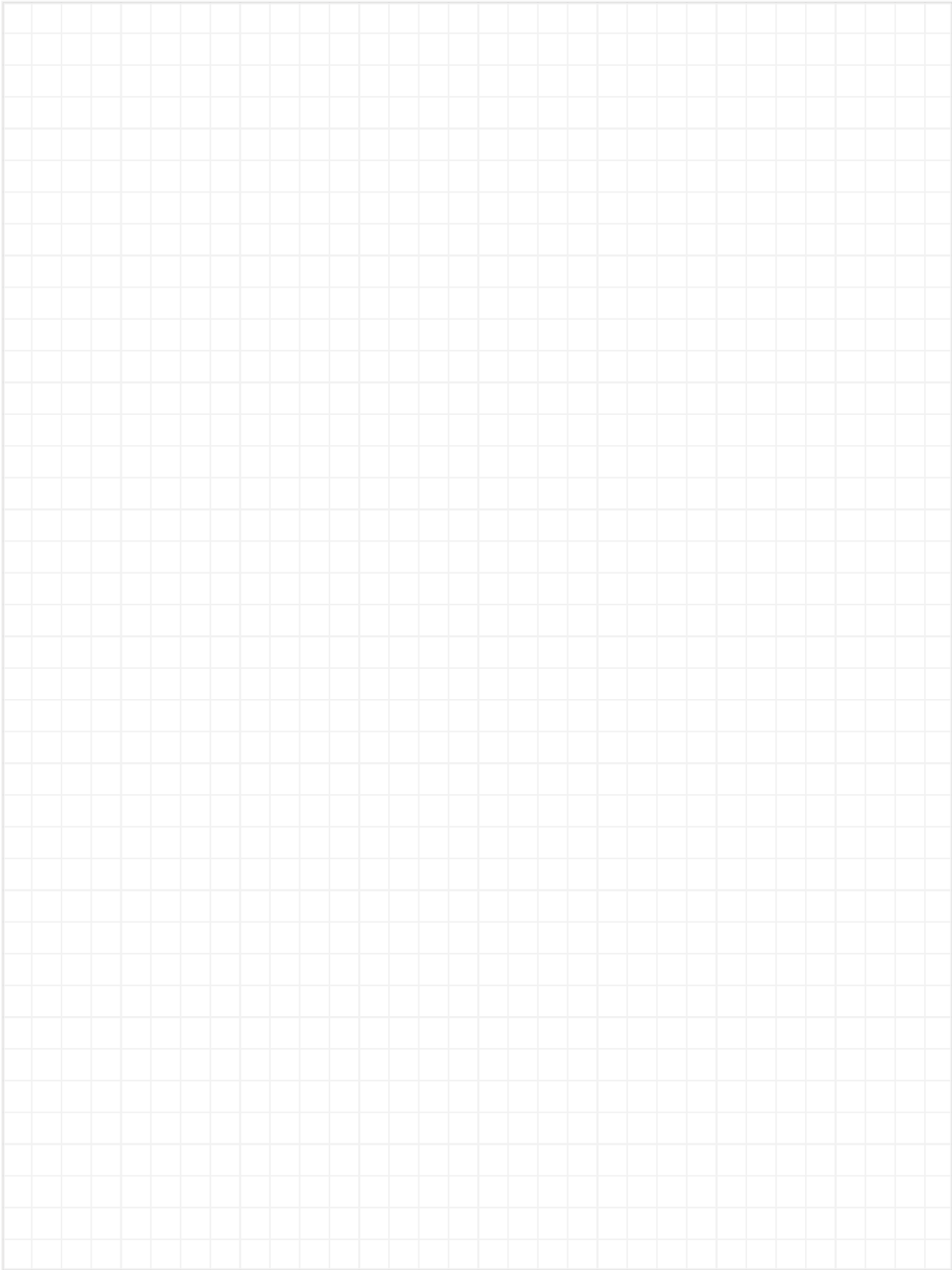
W	Capacity of Water, kg / h
A	Effective Discharge Area, mm ² Low Lift Type : A = TT DL D = Diameter of Disc Seat (mm) L = Lift (mm)
Kd	Coefficient of Discharge, 0.65
Kp	The factor of Over Pressure, 0.6 at 10%
P	Set Pressure, kPag
G	Specific Gravity of water
Kw	Back Pressure Correction factor
Kv	Viscosity Correction Factor
Pb	Back Pressure, kPag
Kc	Safety Valve 1.0

Capacity Table of Water

Low Lift Type		(Unit : kg / h)					
Size		15A	20A	25A	32A	40A	50A
Dia. of Disc Seat (mm)		18	18	24	31	38	48
Lift (mm)		0.45	0.45	0.6	0.775	0.95	1.2
0.2	0.02	252	252	449	749	1125	1796
0.3	0.029	304	304	540	902	1355	2163
0.5	0.049	395	395	702	1172	1762	2811
0.7	0.069	469	469	834	1391	2091	3336
0.9	0.088	529	529	942	1571	2361	3768
1	0.098	559	559	994	1658	2492	3976
1.2	0.118	613	613	1090	1819	2734	4363
1.5	0.147	684	684	1217	2031	3052	4870
1.7	0.167	729	729	1297	2165	3253	5190
2	0.196	790	790	1405	2345	3524	5623
2.2	0.216	830	830	1475	2462	3699	5903
2.5	0.245	884	884	1571	2622	3940	6287
2.7	0.265	919	919	1634	2727	4098	6538
3	0.294	968	968	1721	2872	4316	6887
3.2	0.314	1000	1000	1779	2968	4460	7117
3.5	0.343	1046	1046	1859	3102	4662	7439
3.7	0.363	1076	1076	1913	3192	4796	7652
4	0.392	1118	1118	1988	3317	4984	7952
4.2	0.412	1146	1146	2038	3400	5109	8153
4.5	0.441	1186	1186	2108	3518	5286	8435
4.7	0.461	1212	1212	2156	3597	5405	8624
5	0.49	1250	1250	2222	3708	5572	8891
5.2	0.51	1275	1275	2267	3783	5685	9071
5.5	0.539	1311	1311	2331	3889	5844	9325
5.7	0.559	1335	1335	2374	3961	5952	9496
6	0.588	1369	1369	2435	4062	6104	9740
6.2	0.608	1392	1392	2476	4131	6207	9904
6.5	0.637	1425	1425	2534	4228	6353	10137
6.7	0.657	1447	1447	2573	4294	6452	10295
6.9	0.676	1468	1468	2610	4356	6545	10443
7	0.686	1479	1479	2630	4388	6593	10520
7.2	0.706	1500	1500	2668	4451	6689	10672
7.5	0.735	1531	1531	2722	4542	6825	10889
7.7	0.755	1552	1552	2759	4603	6917	11037
8	0.784	1581	1581	2811	4691	7048	11246
8.2	0.804	1601	1601	2847	4750	7138	11389
8.5	0.833	1630	1630	2898	4835	7265	11593
8.7	0.853	1649	1649	2932	4893	7352	11731
9	0.882	1677	1677	2982	4975	7476	11929
9.2	0.902	1696	1696	3015	5031	7560	12063
9.5	0.931	1723	1723	3064	5112	7681	12256
9.7	0.951	1741	1741	3096	5166	7763	12387
9.9	0.97	1759	1759	3127	5218	7840	12510
10	0.98	1768	1768	3143	5244	7880	12574
10.2	1	1786	1786	3175	5298	7960	12702
Kg / cm ² g MPag		-					
Pressure		-					

NOTE





CERTIFICATE

ISO 9001:2015

ISO 14001:2015

ISO 45001:2018

ASME U, PP Stemp

EAC : RUSSIA TRCU

API 6D / 600 By KSM



CE: GLOBE CONTROL VALVE

CE : BALL VALVE

Research Institute

Venture Company

Material · Part · Equipment

Rooting Enterprise

